



**JURISDICTION AND VENUE**

4. This action arises under the patent laws of the United States, Title 35 of the United States Code, and, accordingly, jurisdiction is conferred on this Court pursuant to 28 U.S.C. §1338(a).

5. This Court has personal jurisdiction over Anvil, including because, on information and belief:

- a. Anvil has sold and offered to sell its products, including the infringing products, in this district;
- b. Anvil owns and operates a manufacturing plant in Columbia, Lancaster County, Pennsylvania, within this district, and various Anvil products, including the infringing products, are manufactured at that plant; and
- c. One of Anvil's four regional distribution centers for the United States is also located in Columbia, Pennsylvania, and the infringing products have been and/or will be distributed by Anvil through that distribution center.

6. Venue is proper in this district pursuant to 28 U.S.C. §§ 1391(b) & (c) and 1400(b).

**VICTAULIC'S PATENTS-IN-SUIT**

7. U.S. Patent No. 7,086,131 (the "131 Patent") was duly and legally issued by the United States Patent and Trademark Office on August 8, 2006 to Victaulic. A copy of the 131 Patent is attached as Exhibit 1.

8. U.S. Patent No. 7,712,796 (the "796 Patent") was duly and legally issued by the United States Patent and Trademark Office to Victaulic on May 11, 2010. A copy of the 796 Patent is attached as Exhibit 2.

9. By assignment, Victaulic is the sole owner of the entire right, title and interest in and to the 131 and 796 Patents and has the exclusive right to sue for infringement of the 131 and 796 Patents.

10. Victaulic has complied with the patent marking statute, 35 U.S.C. § 287.

**THE INFRINGING PRODUCT - ANVIL'S SLIDELOK COUPLING**

11. Anvil's SlideLok coupling, described in more detail in the brochure attached as Exhibit 3, comprises two coupling segments held together by two pairs of bolts and nuts, with an inner gasket.

12. The coupling segments fit around two cylindrical pipe elements to be joined together. The coupling segments deform when installed on a pipe joint in accordance with Anvil's instructions, to conform to the outer surfaces of the pipe elements forming the joint.

13. Anvil is selling, offering for sale, promoting, advertising and marketing its SlideLok coupling in this district and throughout the United States.

14. For example, Exhibit 3 instructs customers and installers that the SlideLok coupling is installed “by simply lubricating the exposed gasket surfaces, and then sliding the coupling completely over the pipe end. . . . Installation is completed by bringing the adjacent component up to the pipe end, and sliding the coupling back over both pipe ends. The joint is finished by tightening the two bolts with an impact wrench or socket wrench.” Anvil touts the advantages of this installation method by explaining that “[t]his method eliminates the possibility of the coupling falling off of the pipe as the entire SlideLOK coupling rests on the pipe end.”

**COUNT ONE**  
**PATENT INFRINGEMENT - U.S. PATENT NO. 7,086,131**

15. Victaulic incorporates and realleges paragraphs 1 to 14.

16. Anvil has directly infringed the claimed method of the 131 Patent, in violation of 35 U.S.C. § 271(a), by practicing the method using the SlideLok coupling, including in the course of developing, testing, displaying and promoting the SlideLok coupling.

17. Anvil is inducing and contributing to infringement of the 131 Patent, in violation of 35 U.S.C. § 271(b) and (c), including by selling its SlideLok coupling and inducing third parties to use the coupling in a manner that infringes the 131 Patent. Third parties, when they use the SlideLok coupling following the instructions in Exhibit 3 (as well as Anvil website demonstration videos and other instructions), practice the steps of the claimed method of the 131 Patent and thus directly infringe the 131 Patent. The SlideLok coupling is a material part of the claimed method of the 131 Patent, and is not a staple article or commodity of commerce suitable for substantial non-infringing use. Anvil has sold and continues to sell the SlideLok coupling in the United States knowing that the coupling is especially adapted to practice the claimed method of the 131 Patent. For example, as shown in Exhibit 3 and website demonstration videos, Anvil instructs third parties to perform each of the steps recited by the claim of the 131 Patent, including the deformation of the coupling segments to conform to the outer surfaces of the pipe elements when installed. Anvil knew of the 131 Patent when it introduced the SlideLok coupling in the market (in fact, Anvil had requested that the United States Patent Office reexamine the 131 Patent before Anvil introduced the SlideLok coupling in the market), knew that its customers would infringe the 131 Patent when they install a SlideLok coupling on a pipe joint, and specifically intended for its customers to infringe the 131 Patent by following Anvil's installation instructions.

18. Anvil's infringement of the 131 Patent has been and is willful.

19. Anvil's infringing activities have caused, and are continuing to cause, substantial damage to Victaulic in an amount to be proven at trial.

20. On information and belief, Anvil will continue to infringe the 131 Patent unless enjoined by this Court.

**COUNT TWO**  
**PATENT INFRINGEMENT - U.S. PATENT NO. 7,712,796**

21. Victaulic incorporates and realleges paragraphs 1 to 14.

22. Anvil has directly infringed the 796 Patent, in violation of 35 U.S.C. § 271(a), including by making, using, offering for sale, and selling the SlideLok coupling and in the course of developing, testing, displaying and promoting its SlideLok coupling.

23. Anvil is inducing and contributing to infringement of the 796 Patent, in violation of 35 U.S.C. § 271(b) and (c), including by selling its SlideLok coupling and inducing third parties to use the coupling in a manner that infringes the 796 Patent. Third parties, when they use the SlideLok coupling following the instructions in Exhibit 3 (as well as Anvil website demonstration videos and other instructions), directly infringe the claims of the 796 Patent. The SlideLok coupling has arcuate surfaces with the characteristics taught by the 796 Patent. The SlideLok coupling is a material part of the invention recited by claims of the 796 Patent, and is not a staple article or commodity of commerce suitable for substantial non-infringing use. Anvil has sold and continues to sell the SlideLok coupling in the United States knowing that the coupling is especially adapted to practice the invention recited by the 796 Patent claims. For example, as shown in Exhibit 3 and website demonstration videos, Anvil instructs third parties to use the SlideLok coupling in combination with a pair of pipe elements as recited in the 796 Patent claims. Anvil knew of the 796 Patent when it introduced the SlideLok coupling in the market (in fact, Anvil had requested that the United States Patent Office reexamine the 796

patent before Anvil introduced the SlideLok coupling in the market), knew that its customers would infringe the 796 Patent when they install a SlideLok coupling on a pipe joint, and specifically intended for its customers to infringe the 796 Patent by following the installation instructions in Exhibit 3.

24. Anvil's infringement of the 796 Patent has been and is willful.

25. Anvil's infringing activities have caused, and are continuing to cause, substantial damage to Victaulic in an amount to be proven at trial.

26. On information and belief, Anvil will continue to infringe the 796 Patent unless enjoined by this Court.

WHEREFORE, Victaulic prays that this Court:

A. Adjudge and decree that Anvil has directly and/or indirectly infringed one or more claims of the 131 and 796 Patents;

B. Preliminarily and permanently enjoin Anvil, its officers, agents, servants, employees, attorneys, and those persons in active concert or participation with it who receive actual notice of the order by personal service or otherwise, from continuing to infringe the 131 and 796 Patents;

C. Order Anvil to pay damages sustained by Victaulic by reason of its infringement of the 131 and 796 Patents, together with interest and costs;

D. Find that this is an exceptional case and order that the damages award be trebled pursuant to 35 U.S.C. § 284;

E. Award Victaulic attorneys' fees pursuant to 35 U.S.C. § 285 and/or any other applicable provision of law; and

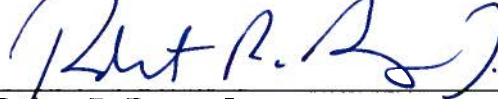
F. Award Victaulic such further relief as this Court deems just and proper.

**JURY DEMAND**

Victaulic hereby demands, pursuant to Fed. R. Civ. P. 38, a trial by jury of all issues so triable.

Dated: October 26, 2012

Respectfully submitted,



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## **Exhibit 1**





US007086131B2

(12) **United States Patent**  
**Gibb et al.**

(10) **Patent No.:** **US 7,086,131 B2**(45) **Date of Patent:** **Aug. 8, 2006**

(54) **DEFORMABLE MECHANICAL PIPE COUPLING**

(75) **Inventors:** John Gibb, Beeton (CA); Douglas R. Dole, Whitehouse Station, NJ (US); Michael S. Pigott, Bluffton, SC (US)

(73) **Assignee:** Victaulic Company, Easton, PA (US)

(\*) **Notice:** Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 0 days.

(21) **Appl. No.:** 11/124,781

(22) **Filed:** May 9, 2005

(65) **Prior Publication Data**

US 2005/0253380 A1 Nov. 17, 2005

**Related U.S. Application Data**

(60) **Provisional application No.** 60/571,596, filed on May 14, 2004.

(51) **Int. Cl.**  
**B21D 39/04** (2006.01)

(52) **U.S. CL.** 29/282

(58) **Field of Classification Search** 285/110, 285/111, 112, 364, 420, 411; 29/282  
See application file for complete search history.

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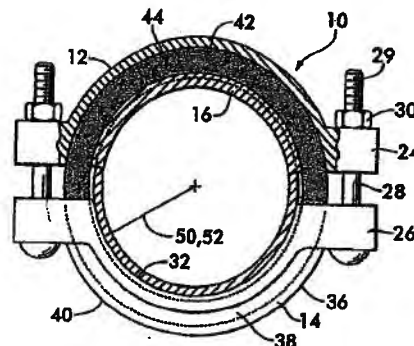
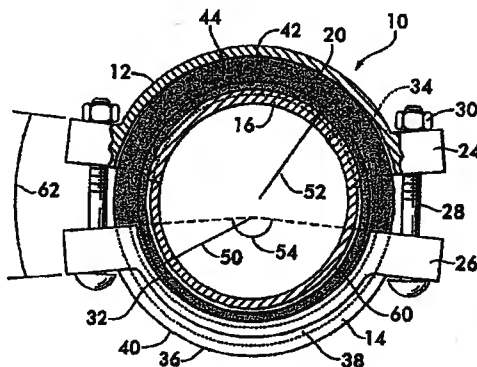
*Primary Examiner*—David Bochna

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(57) **ABSTRACT**

A deformable mechanical pipe coupling is disclosed. The coupling has a plurality of interconnectable segments that straddle the ends of pipe elements to be joined. The segments have arcuate surfaces that engage outer surfaces of the pipe elements. The outer surfaces subtend an angle of less than 180° and have radii of curvature greater than the radii of curvature of the pipe element outer surfaces. The segments have adjustably tightenable connection members for connecting the segments to one another. When the connection members are tightened, the arcuate surfaces deform and conform to the radius of curvature of the outer surface of the pipe elements.

1 Claim, 14 Drawing Sheets



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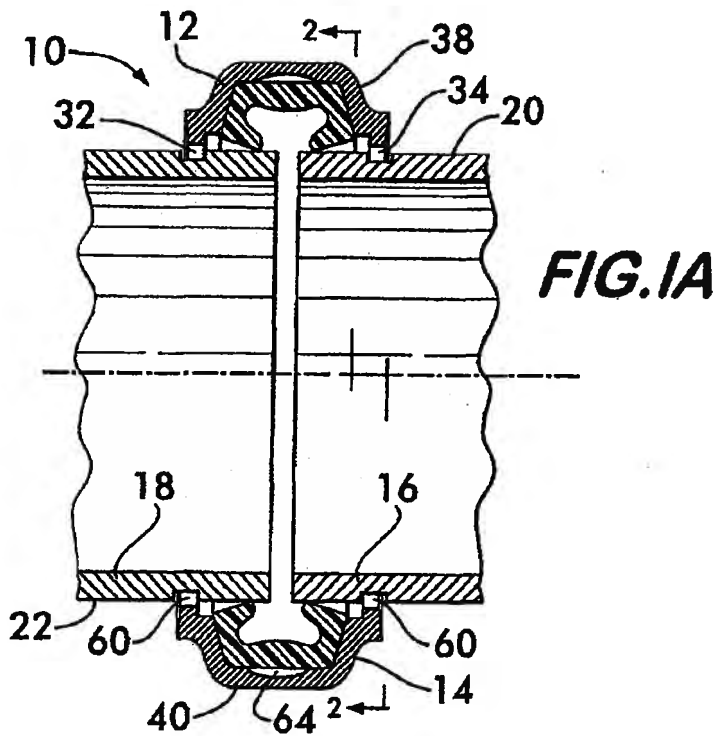
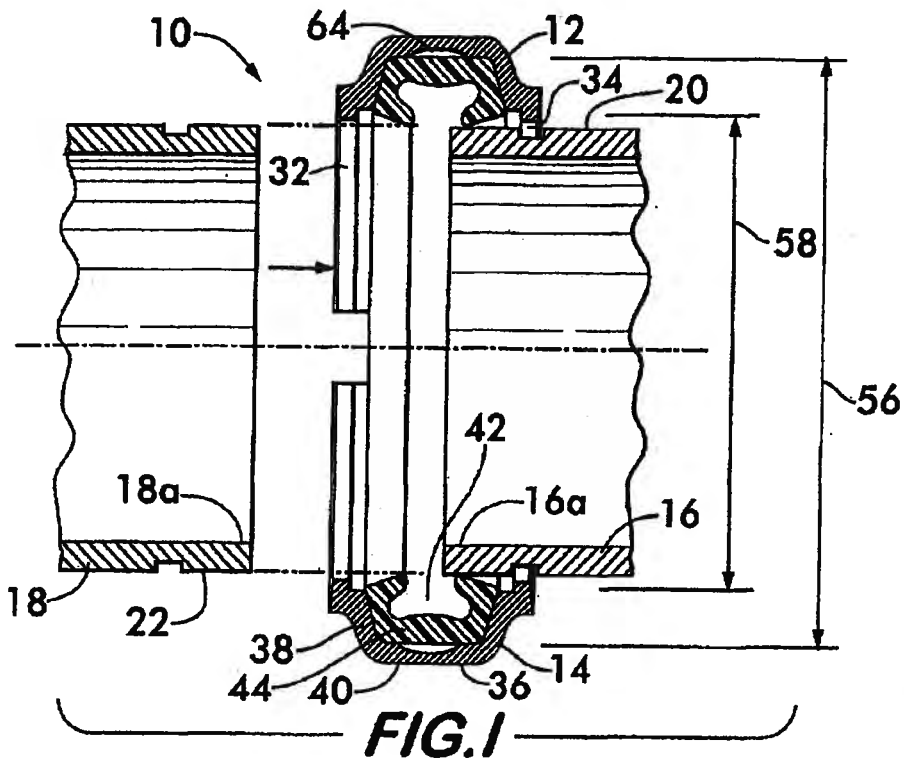
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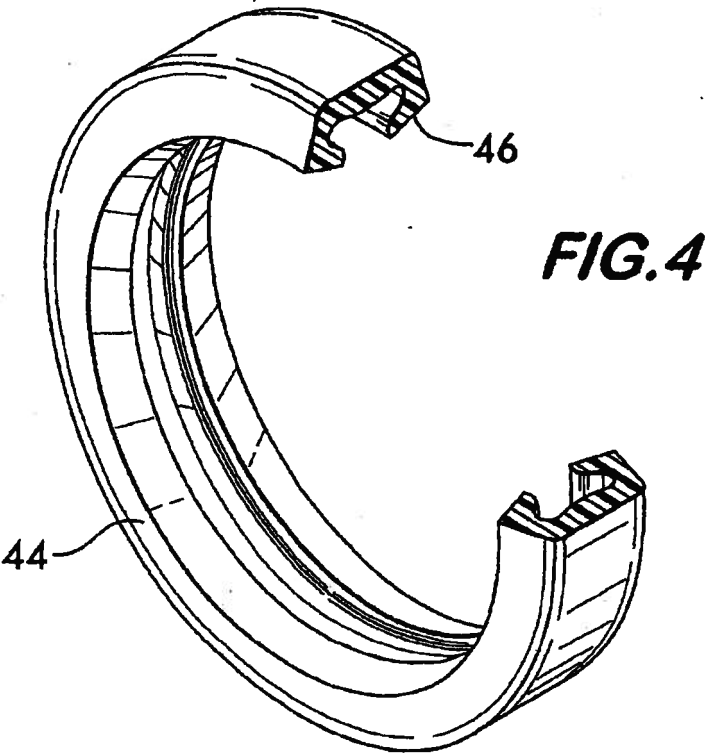
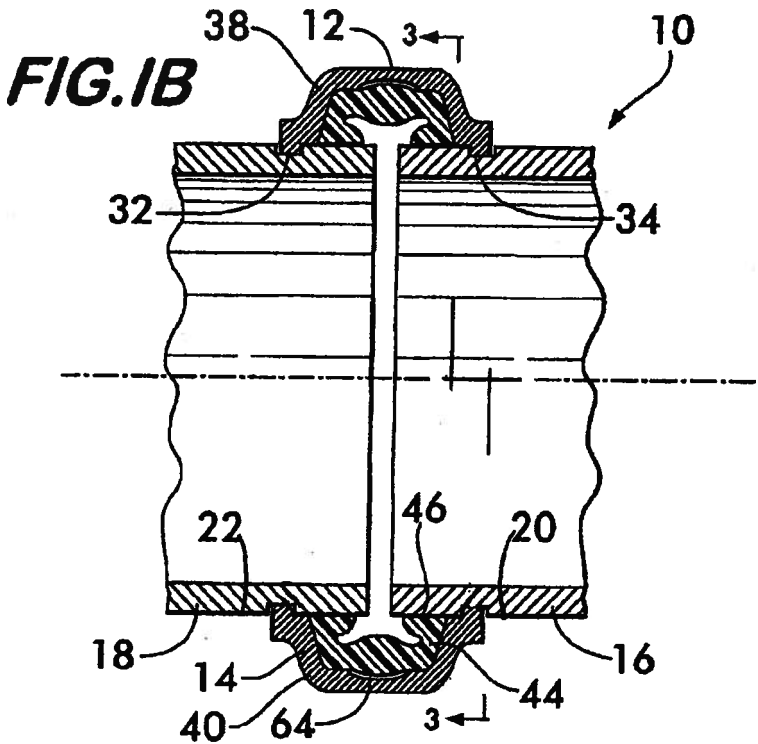


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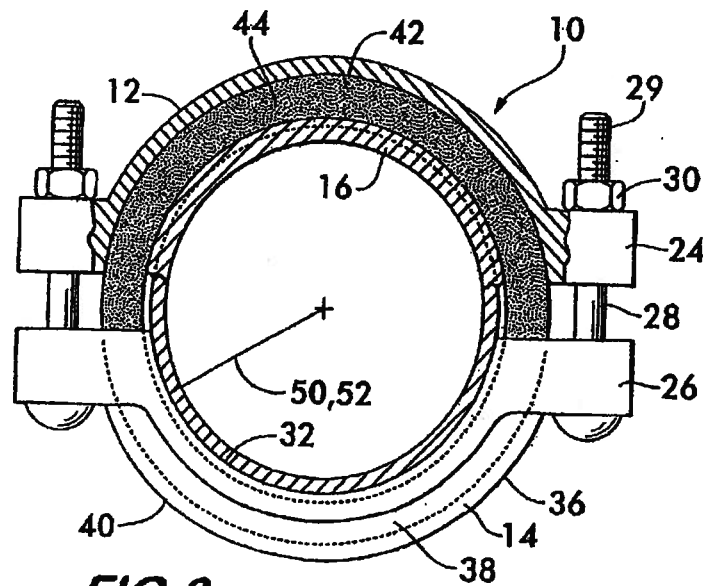
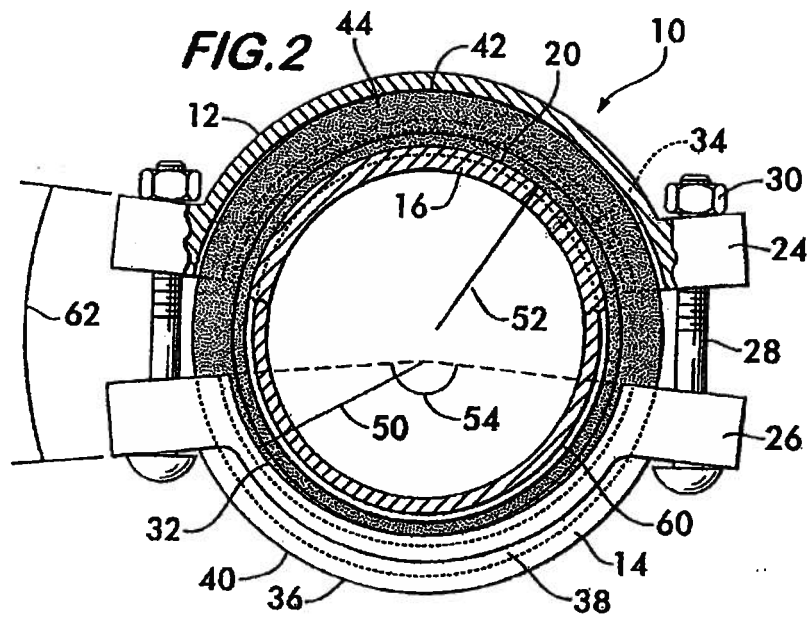


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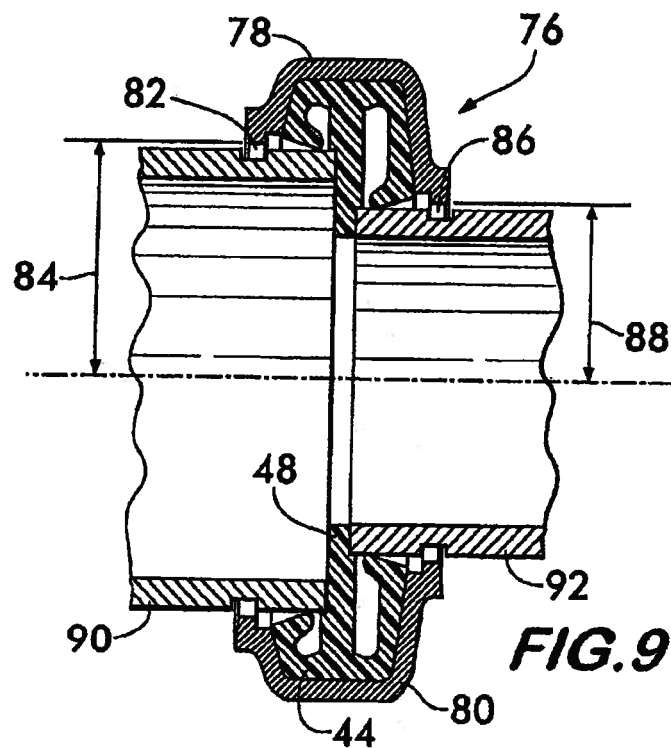
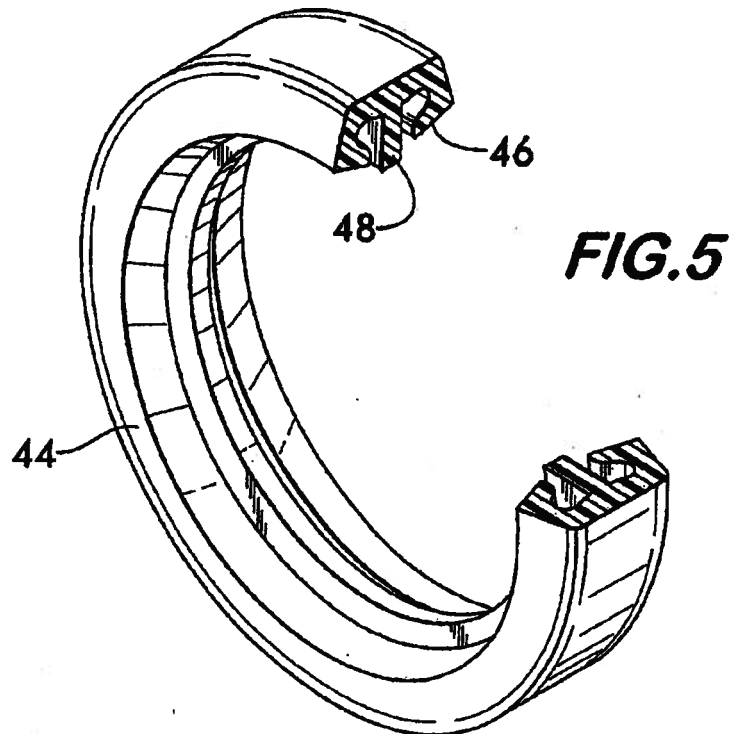


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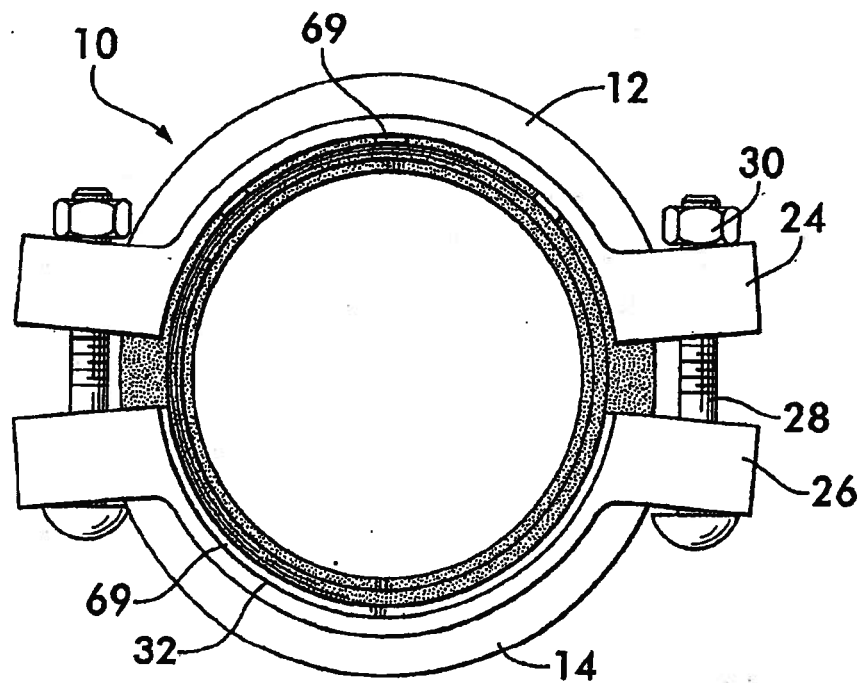
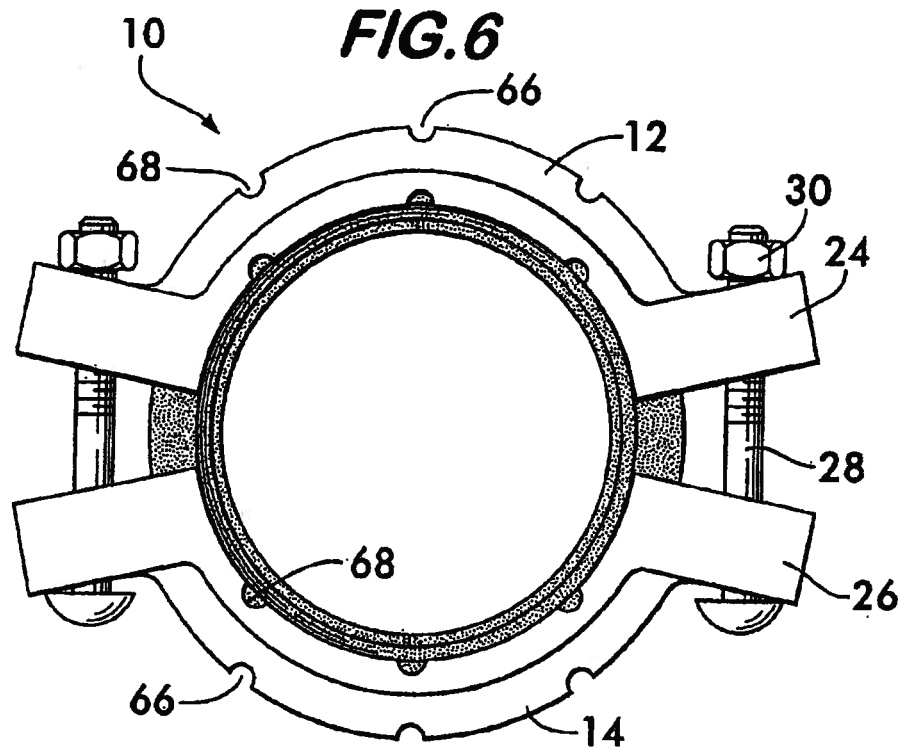


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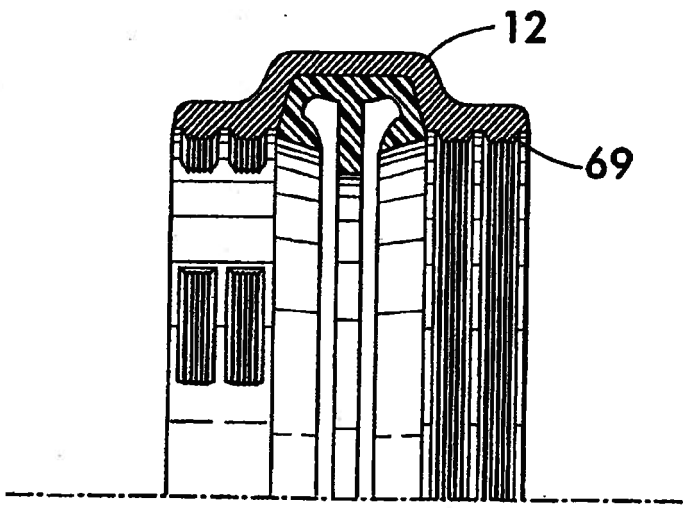
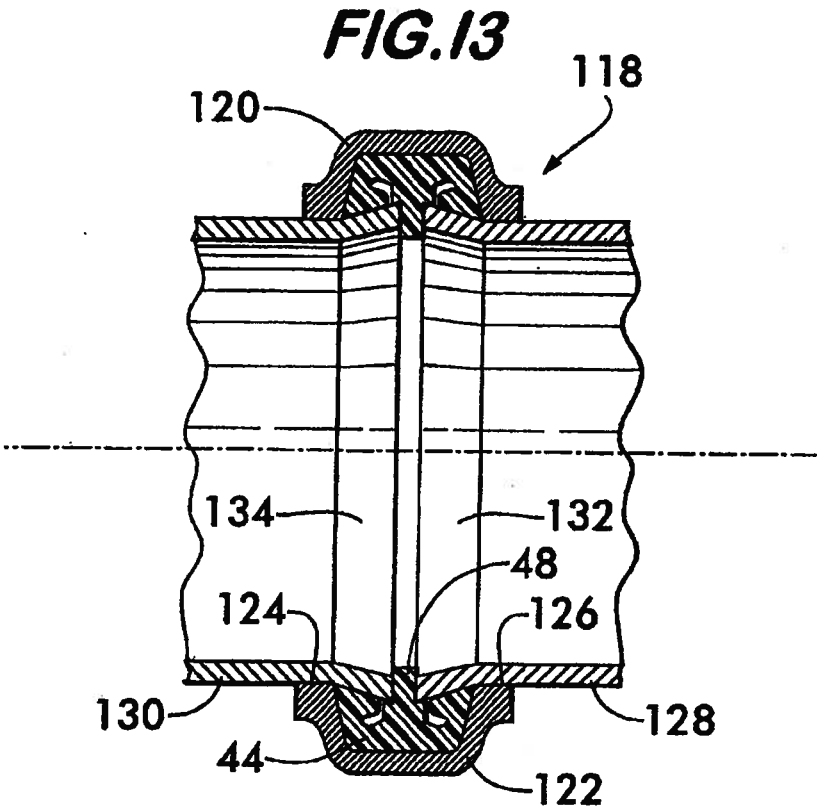
**FIG. 7**

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**FIG. 7A**

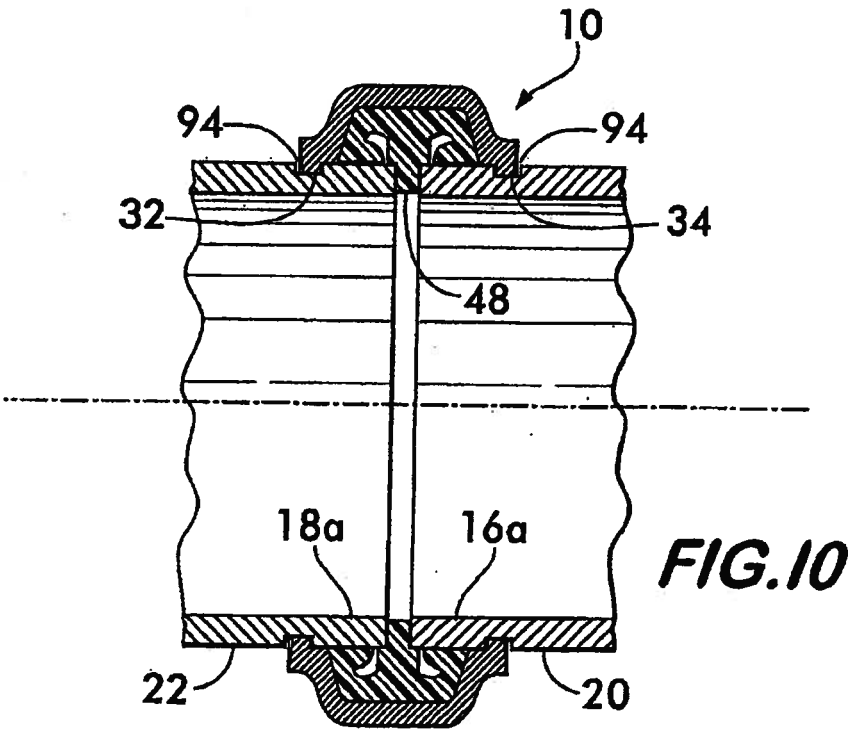
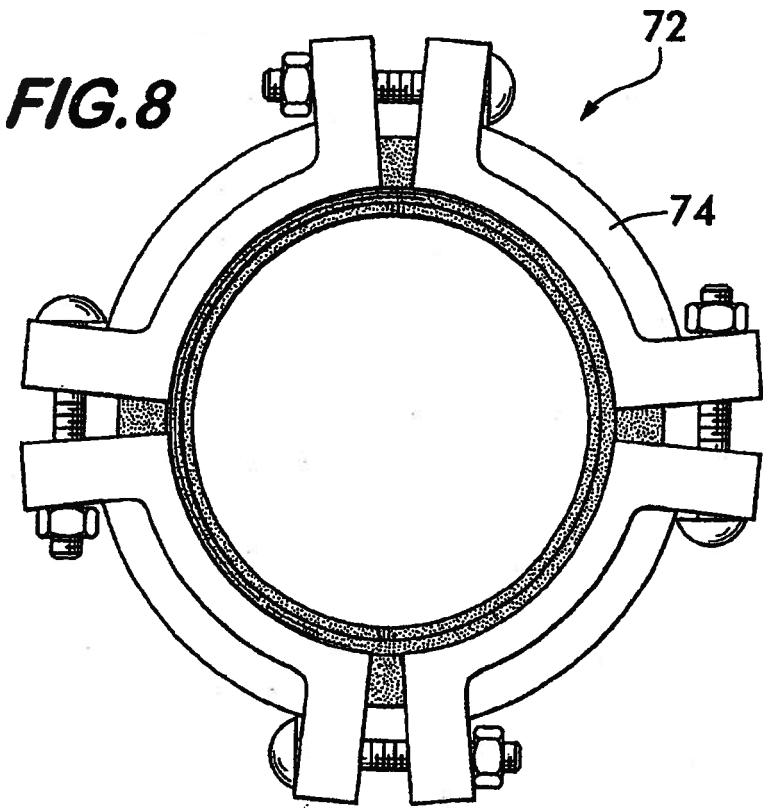


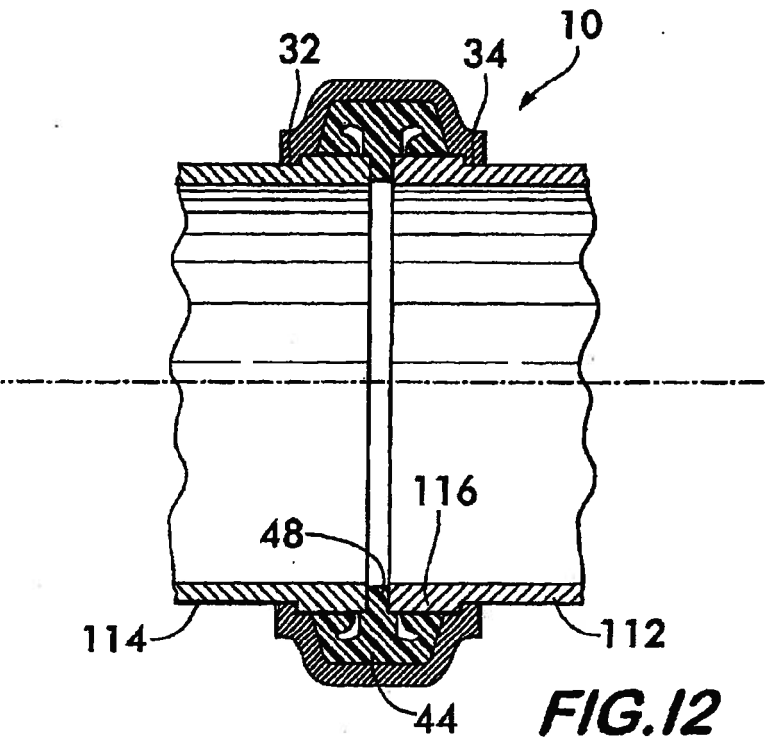
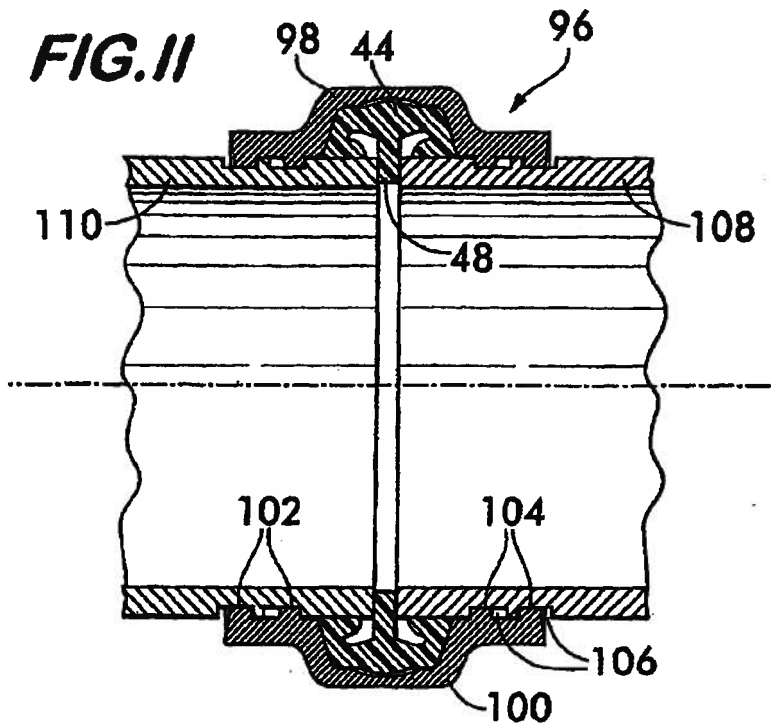
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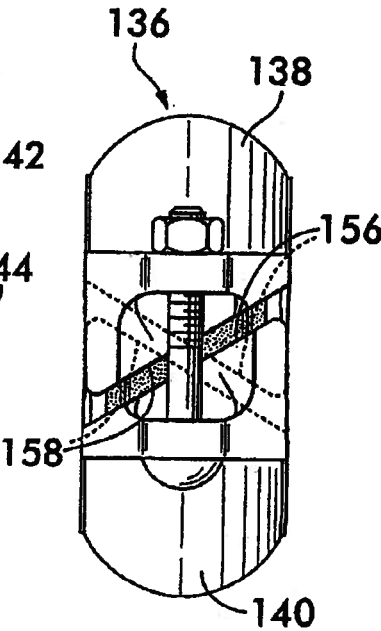
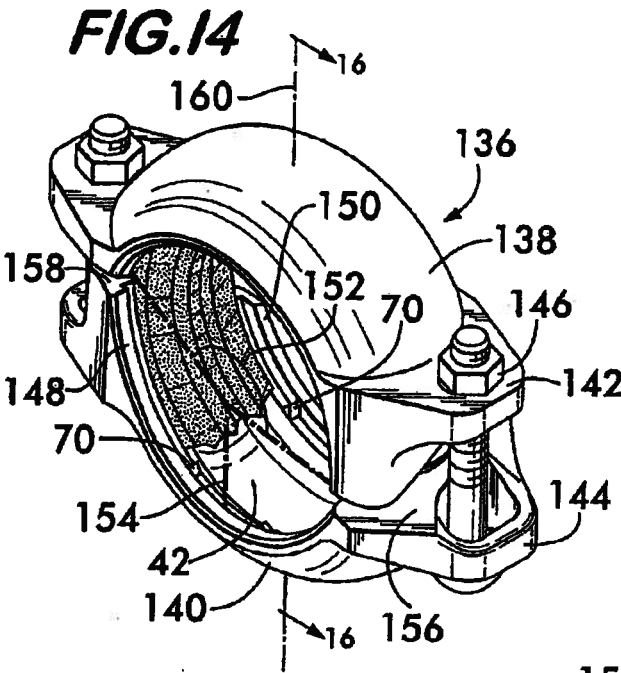


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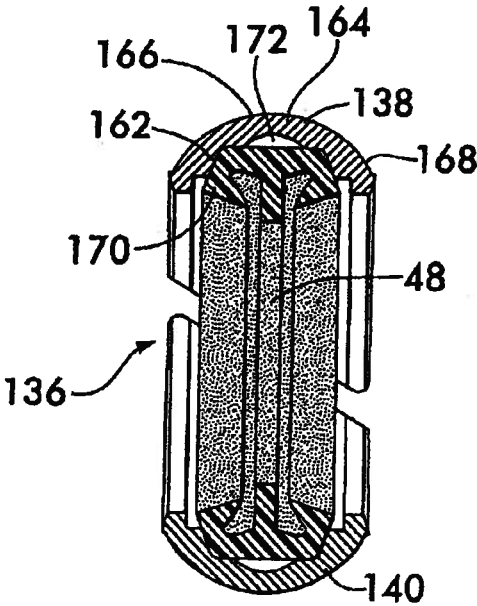
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**FIG. 15**



**FIG. 16**

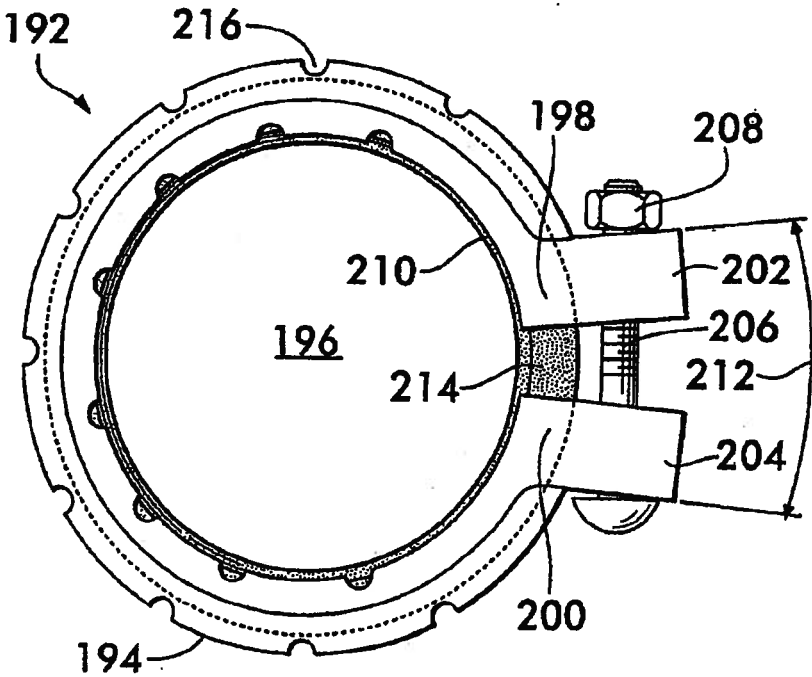
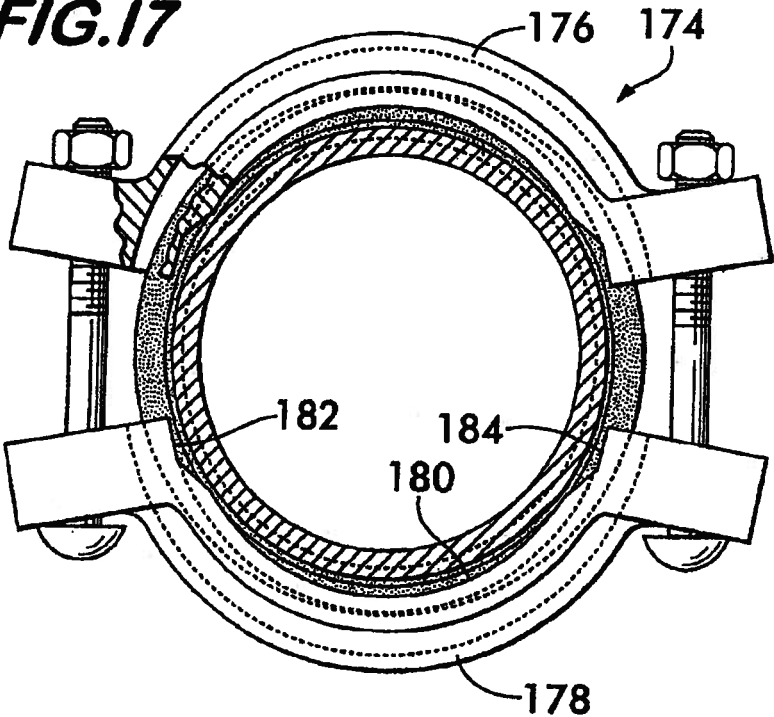
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**FIG. 17**



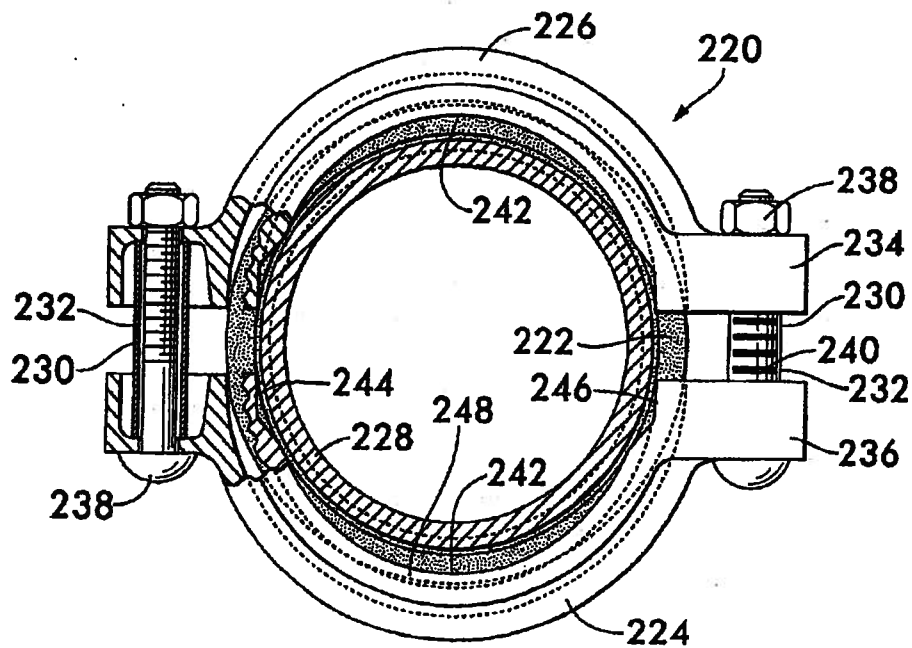
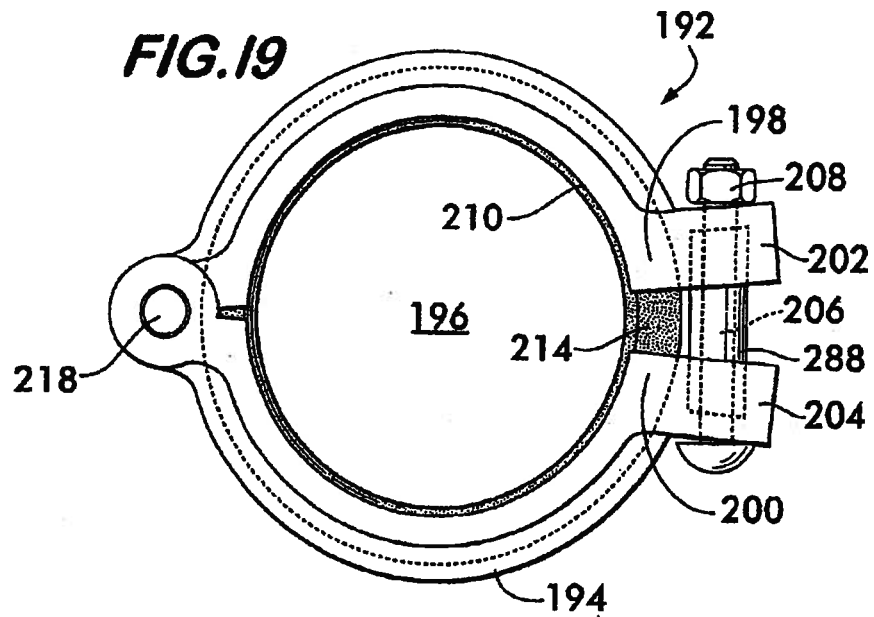
**FIG. 18**

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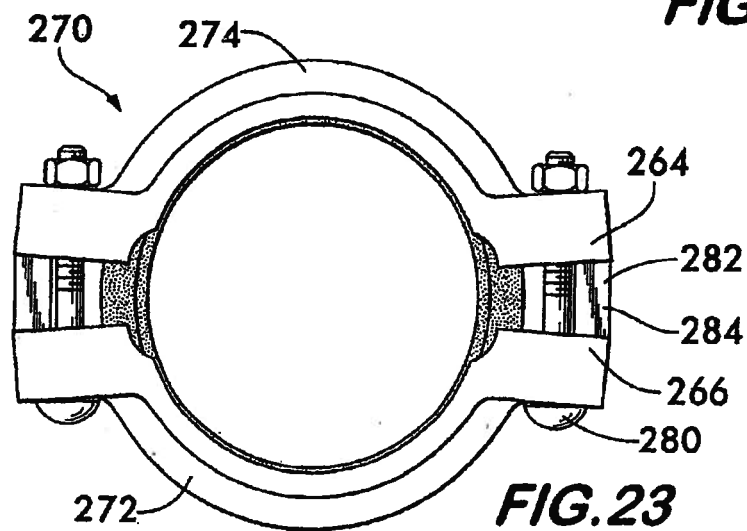
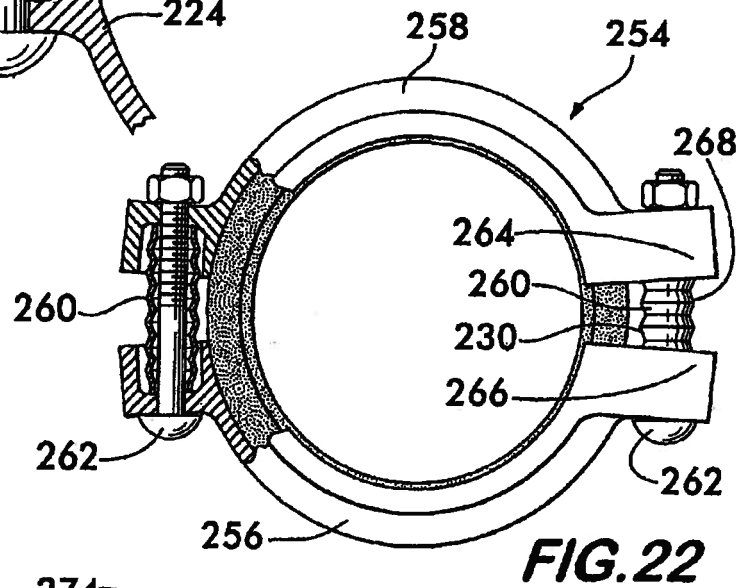
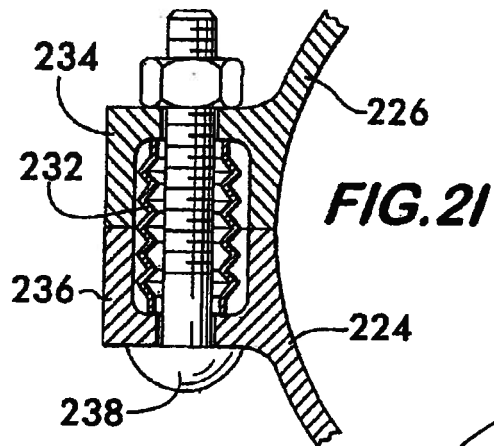
**FIG. 20**

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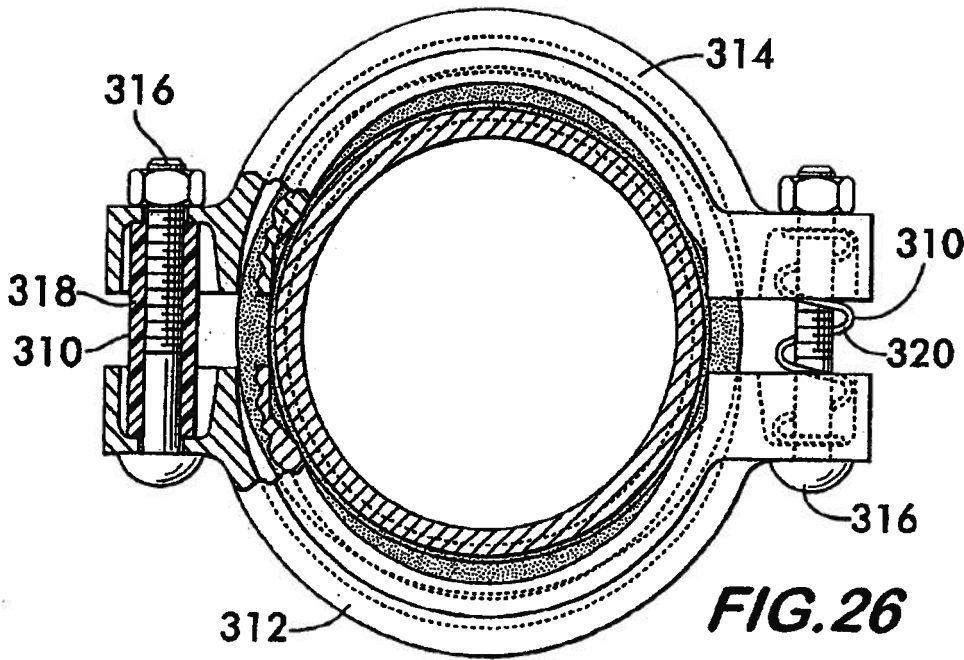
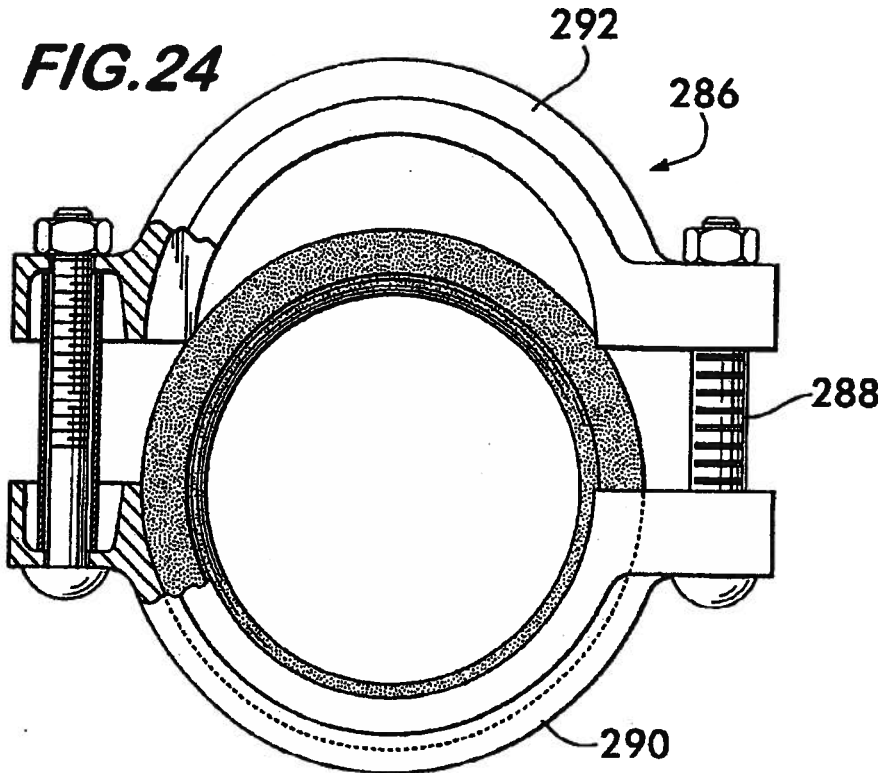


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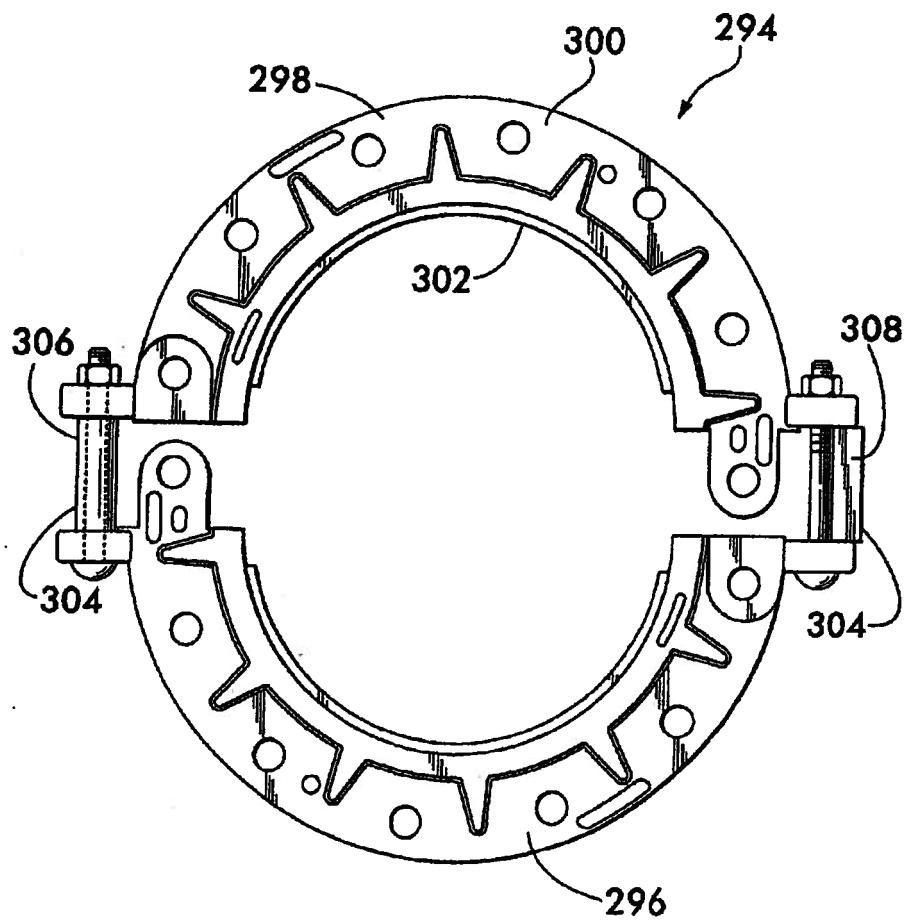


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**FIG. 25**



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1

**DEFORMABLE MECHANICAL PIPE  
COUPLING****FIELD OF THE INVENTION**

This invention concerns mechanical pipe couplings that are deformable to conform to pipe elements, allowing the couplings to be pre-assembled and installed as a unit.

**BACKGROUND OF THE INVENTION**

Mechanical couplings for joining pipe elements together end-to-end comprise interconnectable segments that are positionable circumferentially surrounding the end portions of co-axially aligned pipe elements. The term "pipe element" is used herein to describe any pipe-like item or component having a pipe like form. Pipe elements include pipe stock, pipe fittings such as elbows, caps and tees as well as fluid control components such as valves, reducers, strainers, restrictors, pressure regulators and the like.

Each mechanical coupling segment comprises a housing having arcuate surfaces which project radially inwardly from the housing and engage plain end pipe elements or circumferential grooves that extend around each of the pipe elements to be joined. Engagement between the arcuate surfaces and the pipe elements provides mechanical restraint to the joint and ensures that the pipe elements remain coupled even under high internal pressure and external forces. The housings define an annular channel that receives a gasket or seal, typically an elastomeric ring which engages the ends of each pipe element and cooperates with the segments to provide a fluid tight seal. The segments have connection members, typically in the form of lugs which project outwardly from the housings. The lugs are adapted to receive fasteners, such as nuts and bolts, which are adjustably tightenable to draw the segments toward one another.

To ensure a good fit between the couplings and the pipe elements, the arcuate surfaces on prior art couplings have a radius of curvature that is substantially matched to the radius of curvature of the outer surface of the pipe element that it is intended to engage. For couplings used with grooved pipe elements, the radii of curvature of the arcuate surfaces are smaller than the radii of curvature of the outer surfaces of the pipe elements outside of the grooves so that the arcuate surfaces fit within and engage the grooves properly.

This geometrical relation between the arcuate surfaces of the couplings and the outer surfaces of the pipe elements in prior art couplings results in a tedious and time consuming installation process when mechanical couplings are used. Typically, the coupling is received by the technician with the segments bolted together and the ring seal captured within the segments' channels. The technician first disassembles the coupling by unbolting it, removes the ring seal, lubricates it (if not pre-lubricated) and places it around the ends of the pipe elements to be joined. Installation of the ring seal requires that it be lubricated and stretched to accommodate the pipe elements, an often difficult and messy task, as the ring seal is usually stiff and the lubrication makes manual manipulation of the seal difficult. With the ring seal in place on both pipe elements, the segments are then placed one at a time straddling the ends of the pipe elements and capturing the ring seal against them. During placement, the segments engage the seal, the arcuate surfaces are aligned with the grooves, the bolts are inserted through the lugs, the nuts are threaded onto the bolts and tightened, drawing the coupling segments toward one another, compressing the seal and engaging the arcuate surface within the grooves.

2

As evident from the previous description, installation of mechanical pipe couplings according to the prior art requires that the technician typically handle at least seven individual piece parts (and more when the coupling has more than two segments), and must totally disassemble and reassemble the coupling. Significant time, effort and expense would be saved if the technician could install a mechanical pipe coupling without first totally disassembling it and then reassembling it, piece by piece.

**SUMMARY OF THE INVENTION**

The invention concerns interconnectable pipe coupling segments. Each segment is positionable straddling facing end portions of a pair of pipe elements for securing the pipe elements together in end-to-end relationship. The end portions of the pipe elements have an outer surface of substantially cylindrical profile. Each segment comprises a pair of arcuate surfaces adapted to interface with the outer surfaces of the pipe elements. The arcuate surfaces are in spaced apart relation to one another. The arcuate surfaces subtend an angle of less than 180° and having a radius of curvature greater than the radius of curvature of the outer surfaces of the pipe elements. Each segment further includes connection members for adjustably connecting one coupling segment to another. The connection members are adjustably tightenable for drawing the arcuate surfaces of the segments together. The segments are deformable upon adjustable tightening of the connection members so as to conform the curvature of the arcuate surfaces to the outer surfaces of the pipe elements.

Preferably, the segments are substantially elastically deformable, and the arcuate surfaces project radially inwardly from the segments. The connection members comprise a pair of projections extending outwardly from the ends of each of the segments. The projections are adapted to receive fasteners for adjustably connecting the segments to one another, the fasteners being adjustably tightenable for drawing the arcuate surfaces of the segments together into engagement with the outer surfaces of the pipe elements. Preferably, the projections comprise lugs having an aperture therethrough adapted to receive the fastener.

The invention also concerns a pipe coupling positionable straddling facing end portions of a pair of pipe elements for securing the pipe elements together in end-to-end relationship. Again, the end portions of the pipe elements have an outer surface of substantially cylindrical profile. The pipe coupling comprises first and second coupling segments. Each coupling segment has a pair of arcuate surfaces adapted to interface with the outer surfaces of the pipe elements. The arcuate surfaces are in spaced apart relation to one another. The arcuate surfaces subtend an angle of less than 180° and have a radius of curvature greater than the radius of curvature of the outer surfaces of the pipe elements. Each coupling segment has connection members for adjustably connecting one coupling segment to another. The connection members are adjustably tightenable for drawing the arcuate surfaces of the segments together. The segments are deformable upon adjustable tightening of the connection members so as to substantially conform the curvature of the arcuate surfaces to the outer surfaces of the pipe elements.

A pipe coupling also includes a flexible, resilient seal. The seal is preferably a substantially circular ring having an inner diameter sized to receive the pipe elements. The seal is positioned between the arcuate surfaces of the first and second coupling segments. The seal has an outer diameter sized to position the first and second coupling segments in

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spaced apart relation from one another far enough to thereby allow the pipe elements to be inserted between the first and second coupling segments into end-to-end relationship. The seal is deformable when the connection members are adjustably tightened to draw the arcuate surfaces together and conform the curvature of the segments to the outer surfaces of the pipe elements.

Preferably, each of the first and second segments further comprises a pair of angularly oriented surfaces positioned adjacent to each of the connection members. The angularly oriented surfaces on each segment have opposite slopes. The angularly oriented surfaces on one segment are in facing relation with the angularly oriented surfaces on the other segment. The angularly oriented segments engage one another when the segments are drawn together and cause the segments to rotate relatively to one another about an axis substantially perpendicular to the pipe elements. The drawing together and rotation of the segments forces engagement between the arcuate surfaces and the grooves to provide rigidity about all axes of the joint.

In another embodiment, a pipe coupling comprises an arcuate band having first and second ends in substantially facing relation. The ends are spaced apart from one another. The band surrounds and defines a central space. First and second arcuate surfaces are mounted lengthwise along the band on a side thereof. The arcuate surfaces are in spaced relation alongside one another and project substantially radially inwardly into the central space. The arcuate surfaces have a radius of curvature greater than the radius of curvature of the outer surfaces of the pipe elements. The end portions of the pipe elements are insertable into the central space. Connection members are mounted on the first and second ends of the segments. The connection members are adjustably tightenable for drawing the first and second ends toward one another. The band is deformable, allowing the first and second ends to move toward one another upon adjustable tightening of the connection members. The arcuate surfaces are thereby brought into engagement with the outer surfaces of the pipe elements, their curvature substantially conforming to the curvature of the outer surfaces of the pipe elements.

Deformation of the band may be elastic, plastic, or may be facilitated by a hinge positioned between the first and second ends. The hinge allows a first portion of the band to pivot relatively to a second portion of the band for receiving the pipes within the central space.

The invention also includes a method of securing facing end portions of pipe elements together in end-to-end relationship. The method comprises the steps of:

(A) providing a pipe coupling having a plurality of coupling segments attached to one another end-to-end surrounding a central space, the coupling segments having arcuate surfaces adapted to interface with the outer surfaces of the pipes;

(B) inserting the end portions of the pipe elements into the central space; and

(C) deforming the coupling segments so as to conform the curvature of the arcuate surfaces of the coupling segments to the outer surfaces of the pipe elements.

#### BRIEF DESCRIPTION OF THE DRAWINGS

FIGS. 1-1B are longitudinal cross-sectional views of a deformable mechanical pipe coupling according to the invention;

FIGS. 2 and 3 are partial cross-sectional views of the pipe coupling shown in FIG. 1;

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FIGS. 4 and 5 are perspective views, partially cut away, of seals used with pipe couplings according to the invention;

FIGS. 6-7 and 8 are axial views of various pipe coupling embodiments according to the invention;

FIGS. 7A and 9-13 are longitudinal sectional views of pipe coupling embodiments according to the invention;

FIG. 14 is a perspective view of a pipe coupling according to the invention;

FIG. 15 is a side view of the pipe coupling shown in FIG. 14;

FIG. 16 is a cross-sectional view taken at line 16-16 in FIG. 14;

FIG. 17 is an axial view, partially cut away, of pipe coupling embodiment according to the invention;

FIG. 18 is an axial view of a pipe coupling embodiment according to the invention;

FIG. 19 is an axial view of a pipe coupling embodiment according to the invention;

FIG. 20 is an axial view, partially cut away, of a pipe coupling embodiment according to the invention;

FIG. 21 is a partial sectional view of the pipe coupling shown in FIG. 20;

FIG. 22 is an axial view, partially cut away, of a pipe coupling embodiment according to the invention;

FIG. 23 is an axial view of a pipe coupling embodiment according to the invention; and

FIGS. 24-26 are axial views of a pipe coupling embodiments according to the invention.

#### DETAILED DESCRIPTION OF THE EMBODIMENTS

FIGS. 1 and 2 show a pipe coupling 10 according to the invention. Coupling 10 is formed from coupling segments 12 and 14 which are interconnectable with one another to straddle end portions 16a and 18a of pipe elements 16 and 18 to secure the pipe elements together in end-to-end relationship. The end portions of the pipe elements have respective outer surfaces 20 and 22 of substantially cylindrical profile.

Interconnection of the coupling segments 12 and 14 is effected by connection members, preferably in the form of lugs 24 and 26 best shown in FIG. 2. The lugs are preferably positioned at each end of each segment and project outwardly from the segments. Lugs 24 and 26 are positioned in facing relation to one another and adapted to receive fasteners, preferably in the form of bolts 28 and nuts 30 which are adjustably tightenable and cooperate with the lugs 24 and 26 for adjustably connecting the coupling segments to one another as discussed in further detail below.

As best shown in FIG. 1, each segment 12 and 14 comprises a pair of arcuate surfaces 32 and 34. The arcuate surfaces are in spaced apart relation to one another and preferably project radially inwardly toward the pipe elements 16 and 18. The surfaces extend from a housing 36 having sidewalls 38 joined to a backwall 40, the sidewalls and backwall forming a channel 42 that receives a seal 44.

Examples of seals 44 are shown in FIGS. 4 and 5. Seal 44 is preferably a flexible, resilient ring formed from elastomeric material. The seal may have lips 46 that use the internal pressure within the pipes to increase the sealing force between the seal and the outer surfaces 20 and 22 of the pipe elements 16 and 18. As shown in FIG. 5, seal 44 may also have a tongue 48 positioned between the lips 46, the tongue extending circumferentially around the seal and projecting radially inwardly. Tongue 48 provides a stop surface that engages the ends of pipe elements 16 and 18 to

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ensure proper positioning of the seal 44 relatively to the pipe elements as described in detail below. Engagement of the pipe elements with tongue 48 also effects alignment of the arcuate surfaces with the grooves (if present), or with alignment marks on the outside surface of the pipe elements.

As illustrated in FIG. 2, arcuate surfaces 32 and 34 have radii of curvature 50 greater than the radii of curvature 52 of the outer surfaces 20 and 22 of pipe elements 16 and 18. Furthermore, the arcuate surfaces 32 subtend an angle 54 of less than 180°. Angles 54 between about 40° and about 179° are practical. As a result of this arcuate surface geometry, segments 12 and 14 may be pre-assembled separated from one another such that pipe elements 16 and 18 may be inserted directly into the coupling 10 as shown in FIG. 1 without first disassembling the coupling. This feature provides a significant advantage over prior art couplings which must be assembled onto the pipe ends piece by piece. Joining of the pipe ends with a coupling 10 according to the invention proceeds much more smoothly and quickly than with prior art couplings because the technician handles fewer pieces and does not have to thread nuts onto bolts. In the embodiment shown in FIG. 1, the seal 44 has an outer diameter 56 sized to hold the coupling segments 12 and 14 in spaced apart relation sufficient to allow the pipe ends to be inserted as described above. The seal inner diameter 58 is sized to receive the end portions 16a and 18a of the pipe elements simply by pushing the coupling over the pipe element or by inserting the pipe elements into the coupling. Other embodiments having different features for supporting the segments in spaced relation are described below.

After both pipe elements 16 and 18 are inserted into coupling 10 as shown in FIG. 1A, nuts 30 are tightened (see also FIG. 2). The nuts 30 cooperate with their bolts 28 to draw the arcuate surfaces 32 and 34 on segment 12 toward those on segment 14. Tightening of the nuts exerts a force on the lugs 24 and 26 which brings the segments into contact with the pipe elements and causes the segments 12 and 14 to deform such that the radius of curvature 50 of the arcuate surfaces 32 and 34 substantially conforms to the radius of curvature 52 of the pipe elements 16 and 18. This action is illustrated by comparing FIGS. 2 and 3 and 1A and 1B, wherein the gap 60 between the arcuate surfaces and the pipe outer surfaces diminishes as the arcuate surfaces are brought into engagement with the outer surfaces of the pipe ends. Deformation of the segments 12 and 14 is preferably substantially elastic, allowing the segments to spring back substantially to their original shape when the nuts 30 are loosened, thereby permitting the coupling 10 to be reused in the manner according to the invention as described herein. The segments may also be designed to have significant plastic deformation, wherein the deformation imparts a permanent set to the segments. For practical couplings, there will generally be some degree of both plastic and elastic deformation occurring in the segments as a result of tightening the fasteners. Additionally, when the segments 12 and 14 are in the undeformed state (FIG. 2), the lugs 24 and 26 may be angularly oriented in relation to one another. Relative angles 62 up to about 10° are practical. As shown in FIG. 3, the relative angular orientation of the lugs 24 and 26 is reduced as the segments are deformed, and the geometry may be designed such that the lugs are substantially parallel once the arcuate surfaces 32 and 34 substantially conform to the outer surfaces 20 and 22. This is preferred because, when fully tightened, the bolt head and nut will be in substantially flat contact with the lugs, thereby avoiding induced bending moments in the bolts which can cause permanent deformation of the bolts. The seal 44 is also deformed by this

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process, as shown in FIG. 1B, with the lips 46 coming into full engagement with the pipe element outer surfaces 20 and 22. Because the seal 44 is substantially incompressible, it must be provided with space into which it may expand when compressed by the segments. This space is provided by a concavity 64 positioned in the backwall 40 between the sidewalls 38. Concavity 64 may take virtually any practical shape and allows for volume change of the seal when it is heated or exposed to fluids thereby distributing the deformation of the seal more evenly over its circumference and mitigating the tendency of the seal to extrude outwardly from between the segments between the lugs. The concavity also prevents tongue 48, if present, from being forced between the ends of the pipe elements and impede flow therethrough.

As shown in FIGS. 2 and 3, for the preassembled coupling 10, it is advantageous to hold nuts 30 in a position on bolts 28 that will maintain the segments 12 and 14 in the desired spaced apart relation as determined by contact between the segments and the seal 44. This is conveniently accomplished by deforming the threads 29 of bolts 28, preferably by staking. Staking the bolts hinders the rotation of the nuts and prevents them from unscrewing from the bolts under the effect of vibration, for example, during shipping, and keeps the coupling in the preassembled state with all of its parts together prior to installation. The staking is readily overcome when the nuts are tightened with a wrench.

The bending stiffness of the segments may be tuned to control the amount of force necessary to deform them in order to reduce the required assembly torque and mitigate galling between the nut and the lug. As shown in FIG. 6, sections of increased bending flexibility 66 may be formed in the housing 36 of the segments 12 and 14 by reducing the area moment of inertia of the segment. This reduction is preferably achieved by adding one or more cut-outs 68 in either or both the backwall 40 and the arcuate surfaces 32 and 34.

Alternately, as shown in FIG. 7, the segments may have arcuate surfaces 32 and 34 (not shown) comprising inwardly projecting teeth 69. Teeth 69 engage outer surfaces of the pipe elements to provide mechanical restraint, and are especially advantageous when used with plain end pipe elements. Teeth 69 may be substantially continuous, as shown on segment 14, or intermittent, as shown on segment 12. Single teeth, preferable for small couplings, are also feasible. As shown in FIG. 7A, teeth 69 may also be arranged in pairs on opposite sides of the segment to increase the mechanical restraint provided by the coupling.

Although couplings according to the invention are described above as comprised of two segments, this is by way of example only. Couplings with more than two segments are feasible and preferred for larger diameter pipes due to the manufacturing costs, as reducing the size of the segments is economically advantageous. A further advantage is that the spacing between the lugs is reduced, requiring fewer turns of the nut and shorter bolts. Standard depth sockets may thereby be used during installation. FIG. 8 shows an example of a coupling embodiment 72 having four segments 74 similar to those described above.

Couplings have thus far been shown wherein all of the arcuate surfaces have substantially the same radius of curvature. Whereas such a configuration is appropriate for joining pipes having substantially the same diameter to one another, FIG. 9 shows a coupling embodiment 76 for coupling pipe elements of different diameters. Coupling 76 is formed of two segments 78 and 80 (although it may have more than two segments). Each segment has a first arcuate



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surface 82 having a first radius of curvature 84, and a second arcuate surface 86 having a second radius of curvature 88 smaller than the first radius of curvature 84. This allows coupling 76 to join a pipe element 90 having a larger diameter to a pipe element 92 having a smaller diameter. Analogous to the couplings described above, the radius of curvature 84 is greater than the radius of curvature of the outer surface of pipe element 90, and the radius of curvature 88 is greater than the radius of curvature of the pipe element 92. This geometric relationship allows the pipe elements 90 and 92 to be inserted into a pre-assembled coupling 76 and achieve the advantages of the invention. The coupling segments 78 and 80 deform upon the application of force by adjustable connection members to conform the radii of curvature to the outer surface of the pipe elements.

In a preferred embodiment, shown in FIG. 10, the inwardly projecting arcuate surfaces 32 and 34 of coupling 10 engage grooves 94 formed in the outer surfaces 20 and 22 of pipe element end portions 16a and 18a. Interaction between the arcuate surfaces 32 and 34 with their respective grooves 94 permits the coupling to provide relatively high end restraint to withstand forces caused by internal pressure or external loads. To obtain higher end restraint, it is found useful to add a second set of arcuate surfaces that interact with a second set of grooves in the pipe elements. This embodiment is illustrated in FIG. 11, wherein a coupling 96 is comprised of segments 98 and 100, each segment having two pairs of arcuate surfaces 102 and 104 that project inwardly from the segments. The arcuate surface pairs are in substantially parallel, spaced relation to one another and engage pairs of grooves 106 in the surfaces of the pipe elements 108 and 110 which they connect together.

In another embodiment, shown in FIG. 12, couplings according to the invention such as 10 may be used with pipe elements 112 and 114 having raised circumferential shoulders 116 that are engaged by the arcuate surfaces 32 and 34 of the segments 12 and 14. Alternately, as shown in FIG. 13, a coupling 118 according to the invention having segments 120 and 122 with respective arcuate surfaces 124 and 126 is used with pipe elements 128 and 130 having flared end portions 132 and 134. Note that in the example embodiments shown in FIGS. 9-13, the seal 44 has the tongue 48 which is effectively used to position the pipe ends within the coupling upon insertion, the tongue acting as a pipe stop to aid in locating the pipe ends at the proper depth within the couplings.

Another coupling embodiment 136 is shown in FIG. 14. Coupling 136 is comprised of two segments 138 and 140 from which lugs 142 and 144 extend, the lugs cooperating with fasteners 146 to act as connection members for adjustably connecting one coupling segment to another. As described above, each segment has a pair of arcuate surfaces 148, 150, each preferably projecting radially inwardly from the segments. The arcuate surfaces subtend an angle 152 less than 180° and have a radius of curvature 154 greater than the radius of curvature of the pipe elements which the coupling is to join together. Anti-rotation teeth 70 are positioned adjacent to the arcuate surfaces and project radially inwardly to engage the pipe elements and provide torsional rigidity.

As best shown in FIG. 14, each segment 138 and 140 has a pair of angularly oriented surface portions 156 and 158 located adjacent to each of the lugs 142 and 144. As illustrated, the slope of surface portion 156 may be opposite to the slope of surface portion 158 on each segment. (Both surfaces could also be sloped in the same direction as well.) This opposite slope relationship between the surfaces on a segment results in surfaces having compatible slopes being

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positioned in facing relation in a pre-assembled coupling as shown in FIG. 15. When the fasteners 146 are tightened, conforming the arcuate surfaces to the outer surfaces of the pipe elements, the angular surface portions 156 and 158 on each segment engage and slide relatively to one another, causing the segments to draw together and rotate relatively to one another in opposite directions about an axis 160 oriented substantially perpendicularly to the axis of the pipe elements being joined. These motions of the segments 138 and 140 causes the arcuate surfaces 148 and 150 to engage grooves in the pipe elements and adds rigidity to all axes of the joint as previously described. For coupling segments having surface portions with the same slopes, the couplings move along the pipe in opposite directions relatively to one another with similar effect.

As shown in cross section in FIG. 16, the segments 138 and 140 forming the coupling 136 have a channel 162 defined by a housing 164. The housing is formed from a back wall 166 and sidewalls 168, and receives a seal 170 which is sized to position the segments 138 and 140 in spaced apart relation so as to allow insertion of pipe elements into the pre-assembled coupling shown in FIG. 14. A concavity 172 is provided in the back wall to provide a space for volume change of the seal when it is heated or exposed to fluids as well as to prevent tongue 48 from being forced between the ends of the pipe elements and impede flow therethrough due to compression of the seal.

In another coupling embodiment, shown in FIG. 17, the coupling 174 again is comprised of at least two coupling segments 176 and 178, each having inwardly projecting arcuate surfaces 180 as described above. However, arcuate surfaces 180 have notches 182 and 184 positioned at opposite ends. The notches 182 and 184 provide clearance at the 3 o'clock and 9 o'clock positions of the coupling where it is most needed to permit pipe elements to be inserted into the pre-assembled coupling 174. The availability of increased clearance at these locations allows the coupling segments 176 and 178 to be spaced closer to one another in the pre-assembled configuration than would be the case if the clearance was not available at the ends of the surfaces. By having the segments of the preassembled coupling closer together, the amount of deformation required to conform the arcuate surfaces to the pipe element outer surface is reduced and thereby the energy required to tighten the fasteners.

Another coupling embodiment 192 according to the invention is shown in FIG. 18. Coupling 192 comprises an arcuate band 194 surrounding a central space 196. Band 194 has opposite ends 198 and 200 positioned in facing relation to one another. Ends 198 and 200 are in spaced relation in the pre-assembled coupling and have connection members mounted thereon, preferably in the form of projecting lugs 202 and 204 adapted to receive a fastener such as bolt 206 and nut 208. The bolt and nut cooperate with the lugs to deform the band 194 and bring the ends 198 and 200 toward one another after pipe elements have been inserted into the central space 196 for coupling in end-to-end relationship. Band 194 has a pair of arcuate surfaces 210, only one of which is visible in the figure. The arcuate surfaces are in spaced relation lengthwise of one another as illustrated in FIG. 10 and described above for other embodiments. The arcuate surfaces 210 have a greater radius of curvature than the outer surface of the pipe ends that the coupling is to join together. This geometric configuration, and the separation of the ends 198 and 200 allows the pipe elements to be inserted into central space 196. Upon tightening of the nut 208 the band 194 is deformed such that the radius of curvature of the arcuate surfaces 210 are forced to conform with the radius

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of curvature of the outside surface of the pipe elements which they engage. Note that in the preassembled state, projecting lugs 202 and 204 are preferably angularly oriented with respect to one another, having a relative angle 212 up to about 20°. Tightening of the fastener draws the lugs toward each other, and results in decreasing the relative angle 212, preferably to the point wherein the lugs are substantially parallel to one another. This is particularly advantageous for a flexible coupling which does not depend upon the pipe elements to form a reaction point to cause the deformation in combination with the bolts, the friction incurred at the reaction points inhibiting flexibility.

Coupling 192 includes a seal 214 positioned within the band 194 between the arcuate surfaces 210. Seal 214 may be similar to those illustrated in FIGS. 4 and 5 and sized to receive the pipe elements for creating a fluid tight seal when the band is deformed.

The bending flexibility of coupling 192 may be adjusted by reducing the area moment of inertia of band 194. Such adjustments may be effected by positioning cut-outs 216 in the band. Alternately, as shown in FIG. 19, a hinge 218 may be provided between the ends 198 and 200. Hinge 218 is preferably positioned equidistant from the ends of the band and provides infinite bending flexibility, reducing the torque needed on the fastener to draw the ends 198 and 200 toward one another. The band 194 will still deform as the arcuate surfaces 210 engage the outer surfaces of pipe elements to conform the radii of the surfaces with that of the pipe element outer surfaces. When the hinge is present, the seal 214 is sized so as to maintain the lugs 202 and 204 in spaced relation so that pipe elements may be inserted. For both the hinged and hingeless versions of the coupling described above, the arcuate surfaces preferably project radially inwardly from the band and may have different radii of curvature from each other, as illustrated in FIG. 9, to allow the coupling 192 to be used to join pipes having different diameters.

FIG. 20 illustrates a pre-assembled coupling 220 that does not depend on the seal 222 to maintain its segments 224 and 226 in spaced apart relation and ready to receive pipe elements such as 228. Coupling 220 has spacers 230 that extend between segments 224 and 226 and maintain the segments in spaced apart relation. In this example embodiment, the spacers 230 comprise collapsible tubes 232 that are positioned between facing lugs 234 and 236 that extend from the segments. Tubes 232 are preferably thin walled and circular in cross section and are arranged coaxially surrounding the fasteners 238. The tubes may be made of lightweight metal or a polymer material such as polypropylene and may have score lines 240 in their surface to create weakened regions that facilitate collapse of the tube under compressive loads applied by the fasteners 238. Other materials, such as cardboard and rubber are also feasible. The tubes are designed to be strong enough to support the segments in spaced relation during shipping, handling and installation, but collapse at a predetermined compressive load that a technician may apply, preferably by manually tightening the fasteners with a wrench.

In use, pipe elements to be joined end-to-end are inserted between the segments 224 and 226. Fasteners 238 are then tightened to draw the segments toward each other and into engagement with the pipe elements. Tightening of the fasteners places the tubes 232 under a compressive load, and the tubes buckle and collapse as shown in FIG. 21 when the predetermined load is achieved to allow the segments to move toward one another and engage the pipe elements to effect the joint.

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Spacers positioned between the segments may be used with any type of mechanical coupling. Note that in FIGS. 20 and 21, the segments 224 and 226 have arcuate surfaces 242 with a radius of curvature that is substantially the same as the radius of curvature of the outer surface of pipe element 228 which they are designed to engage. To provide clearance between the pipe element 228 and the segments allowing the pipe element to be inserted into the coupling while still maintaining a reasonable fastener length, notches 244 and 246 are positioned at opposite ends of the arcuate surfaces 242 as best shown in FIG. 20. The notches provide clearance at the 3 o'clock and 9 o'clock positions of the coupling to permit pipe elements to be inserted into the pre-assembled coupling 220.

FIG. 22 illustrates another coupling embodiment 254 having spacers 230 between segments 256 and 258 comprising the coupling. In this example, the spacers 230 comprise tubes 260 again positioned coaxially with fasteners 262 and between facing lugs 264 and 266 projecting from the segments. Tubes 260 have corrugations 268 which facilitate their collapse when compressive load is applied by tightening the fasteners. Note that the segments 256 and 258 are similar to those described above with respect to FIGS. 1 and 2, wherein the arcuate surfaces of the segments have a greater radius of curvature than the pipe elements.

Another example of a spacer for maintaining coupling segments in spaced relation is shown in FIG. 23. Coupling 270 is comprised of segments 272 and 274 having outwardly projecting lugs 266 and 268 positioned in facing relation when the coupling is pre-assembled. The segments are held together by fasteners 280 extending between the lugs. Spacers 282, preferably in the form of block-shaped bodies 284, are positioned between the lugs 266 and 268. The bodies 284 are removable from between the lugs to allow the fasteners to be tightened and draw the segments into engagement with pipe elements being joined.

Bodies 284 may be releasably attached to the segments, for example, held by friction between the lugs 266 and 268. Flexible, resilient materials are particularly advantageous for forming the bodies because bodies made from such materials provide adequate strength and stiffness to maintain the couplings in spaced apart relation during rough handling but may be readily deformed for easy removal as required. If polymer materials are used to form the bodies they may be adhered to the lugs by heat fusing or by adhesives which afford a releasable bond between the bodies and the segments.

FIG. 24 illustrates a non-deformable coupling embodiment 286 that uses spacers 288 to maintain coupling segments 290 and 292 in spaced apart relation so that pipe elements may be inserted between them in the preassembled state shown. Coupling 286 has no notches or other features that provide clearance facilitating inserting pipe elements into end to end relation between the segments, but relies on the spacers to provide sufficient separation for adequate clearance. The spacers 288 may be similar to those described above.

Spacers according to the invention may also be used with various other types of couplings. As shown in FIG. 19, a spacer 288 may be used with the hinged coupling embodiment 192 to keep the lugs 202 and 204 in spaced apart relation so that pipe elements may be inserted. Although a tubular spacer is illustrated, it is understood that any of the spacers described herein are feasible for use with this coupling.

FIG. 25 illustrates an adapter coupling 294 for joining flanged pipe to non-flanged pipe, for example, grooved or

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plain end. Coupling 294 comprises coupling segments 296 and 298, each of which has a radially extending flange 300 on one side and a arcuate surface 302 on the opposite side. Segments 296 and 298 are held in spaced apart relation by spacers 304, which may comprise a collapsible tubular spacer 306 or a removable spacer 308, or other types of spacers described herein.

FIG. 26 illustrates further types of spacer embodiments 310 feasible for maintaining coupling segments 312 and 314 in spaced apart relation. Spacers 310 comprise spring elements which deform, preferably substantially elastically, when subjected to a compression force by the fasteners 316. Spring elements may take any of a number of forms, for example, a rubber cylinder 318 or a coil spring 320. The use of spring elements for spacers allows for fine control of the forces required to draw the segments toward one another and also facilitates the reuse of the couplings when the deformation of the spring elements is substantially elastic.

It is anticipated that deformable couplings may also include features such as the tongue and recess disclosed in U.S. Pat. Nos. 6,170,884 and 6,302,450; outlets incorporated within a segment as disclosed in U.S. Pat. No. 3,362,730; plain end couplings that do not use grooves as disclosed in U.S. Pat. Nos. 2,439,979, 3,024,046, 5,911,446 and

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6,302,450, all of these patents being hereby incorporated by reference herein.

Deformable mechanical pipe couplings according to the invention provide for rapid and sure installation creating a pipe joint while avoiding the need to partially or totally disassemble and then reassemble the coupling and handle the individual piece parts.

What is claimed is:

1. A method of securing facing end portions of pipe elements together in end-to-end relationship, wherein said end portions of said pipe elements have an outer surface of substantially cylindrical profile, said method comprising the steps of:

providing a pipe coupling having a plurality of coupling segments attached to one another end-to-end surrounding a central space, said coupling segments having arcuate surfaces adapted to interface with the outer surfaces of said pipes;

inserting said end portions of said pipe elements into said central space; and

deforming said coupling segments so as to conform the curvature of said arcuate surfaces of said coupling segments to said outer surfaces of said pipe elements.

\* \* \* \* \*

## Exhibit 2





US007712796B2

(12) **United States Patent**  
**Gibb et al.**

(10) **Patent No.:** **US 7,712,796 B2**  
(45) **Date of Patent:** **May 11, 2010**

(54) **DEFORMABLE MECHANICAL PIPE COUPLING**

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(\*) **Notice:** Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 95 days.

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## ABSTRACT

A deformable mechanical pipe coupling is disclosed. The coupling has a plurality of interconnectable segments that straddle the ends of pipe elements to be joined. The segments have arcuate surfaces that engage outer surfaces of the pipe elements. The outer surfaces subtend an angle of less than 180° and have radii of curvature greater than the radii of curvature of the pipe element outer surfaces. The segments have adjustably tightenable connection members for connecting the segments to one another. When the connection members are tightened, the arcuate surfaces deform and conform to the radius of curvature of the outer surface of the pipe elements.

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**Related U.S. Application Data**

(62) Division of application No. 11/124,781, filed on May 9, 2005, now Pat. No. 7,086,131.

(60) Provisional application No. 60/571,596, filed on May 14, 2004.

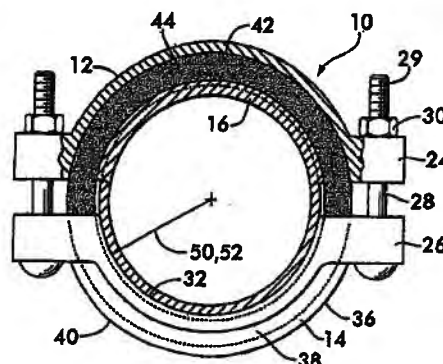
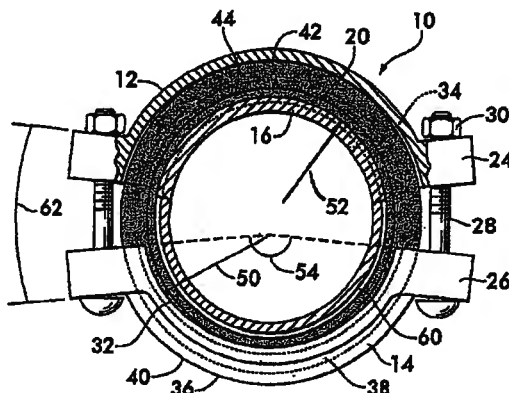
(51) **Int. Cl.**  
*F16L 23/00* (2006.01)

(52) **U.S. Cl.** ..... 285/411; 285/367

(58) **Field of Classification Search** ..... 285/110, 285/111, 112, 364, 420, 411, 367  
See application file for complete search history.

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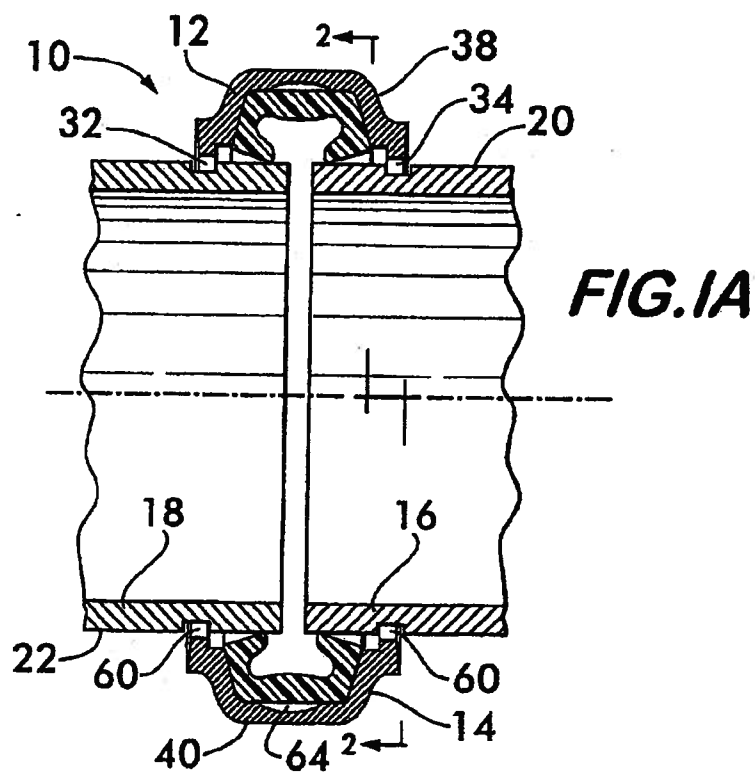
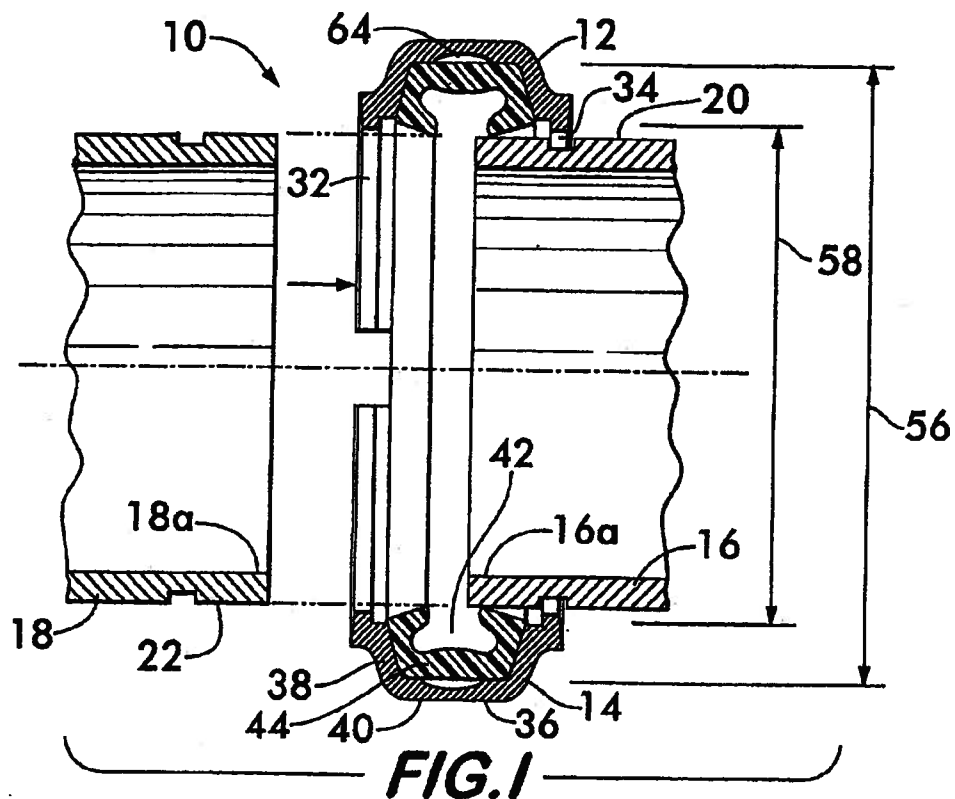
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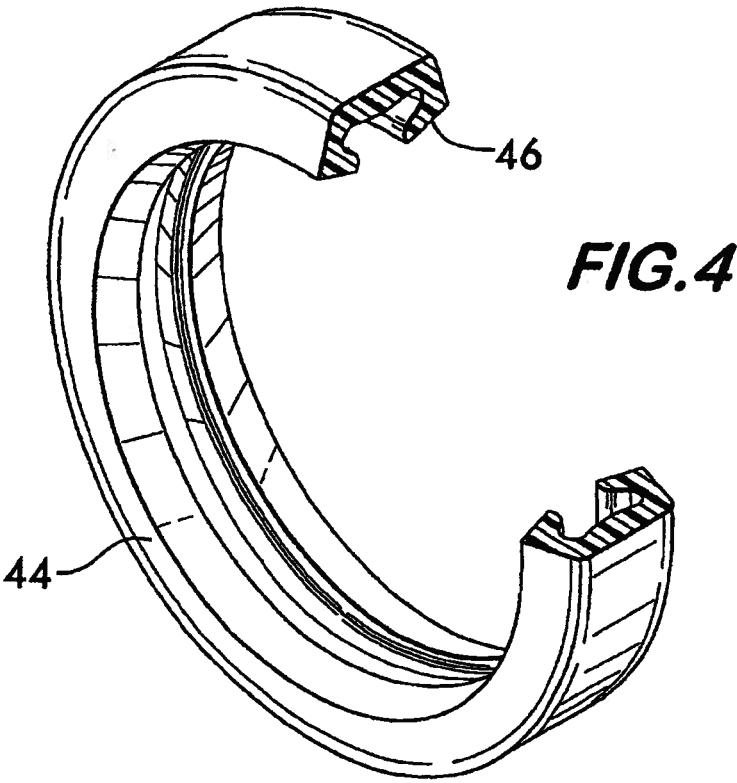
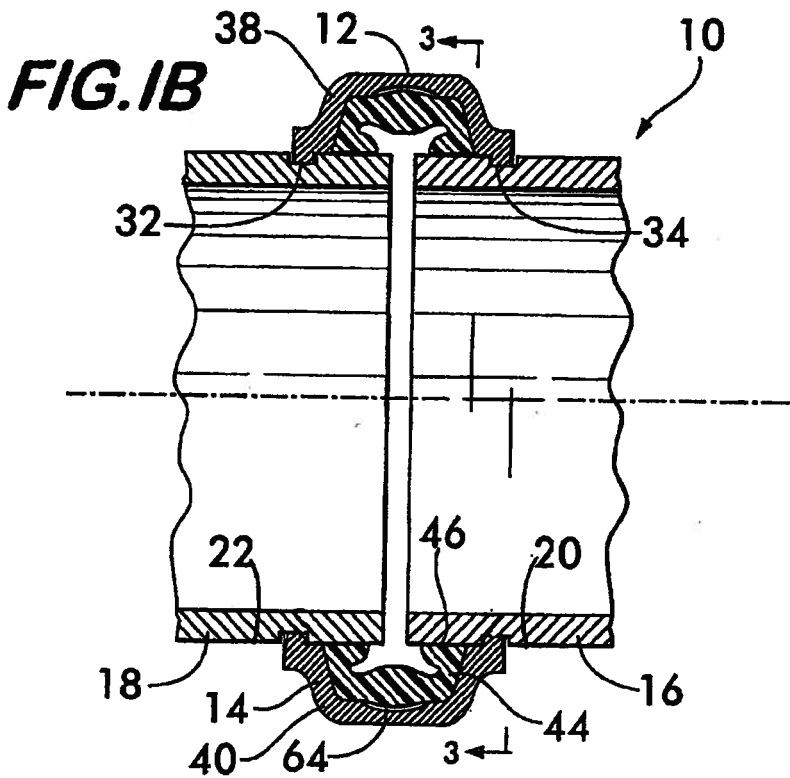
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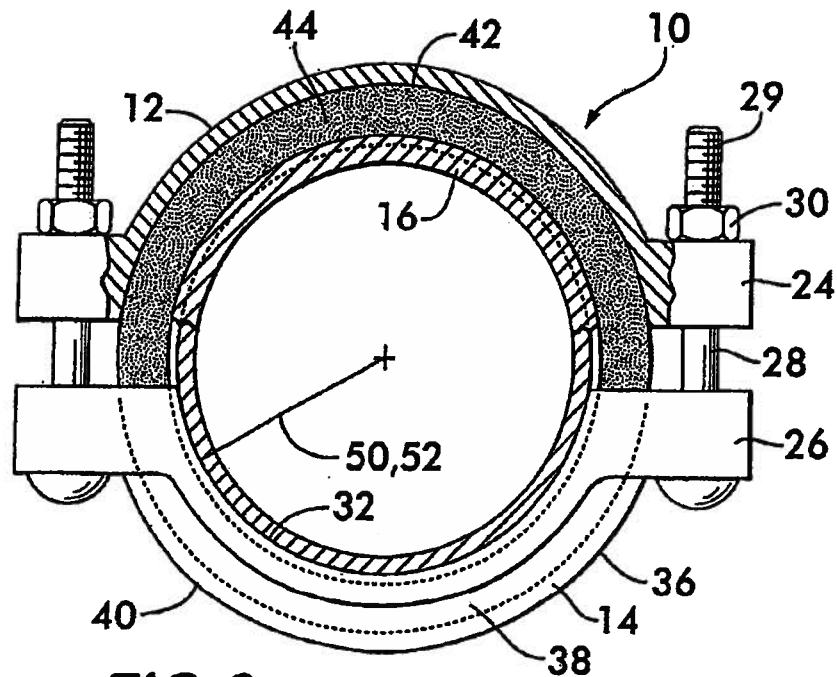
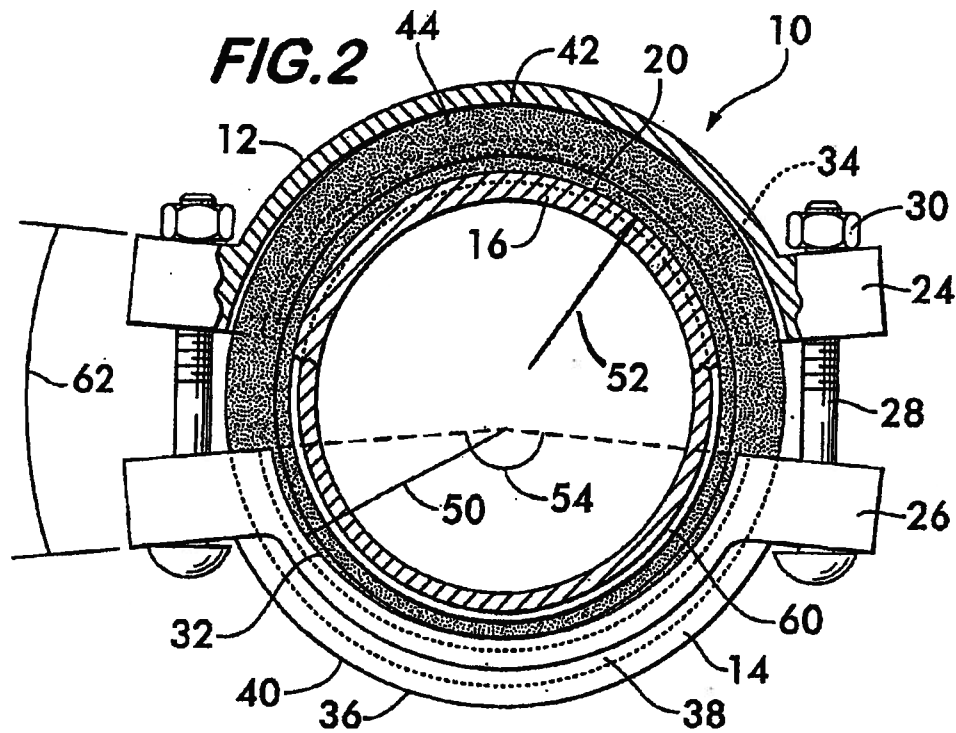


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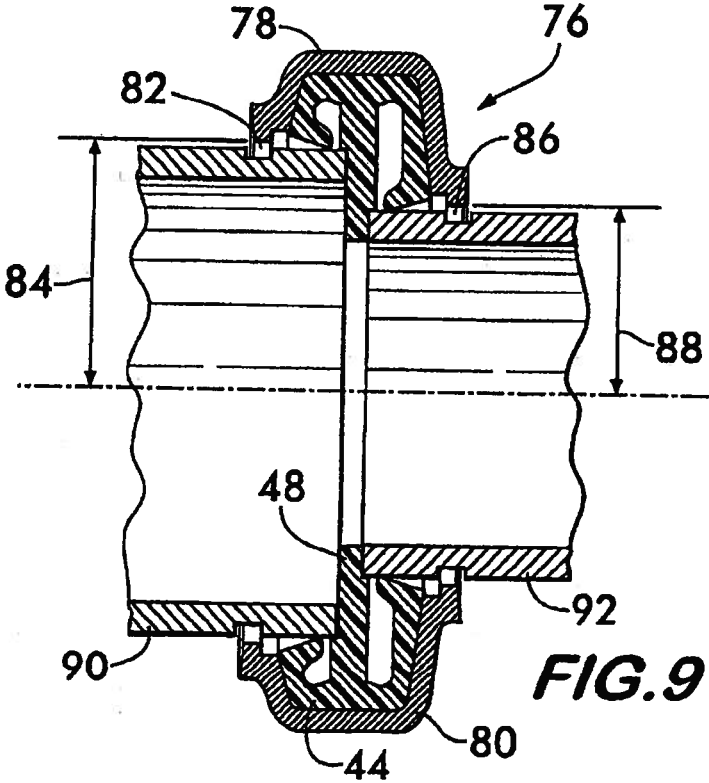
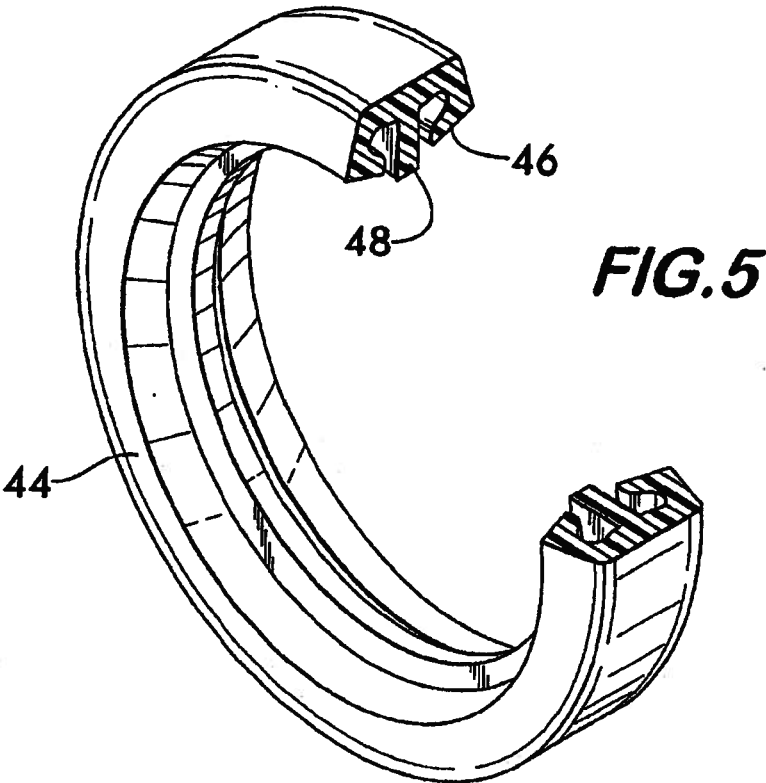
**FIG. 3**

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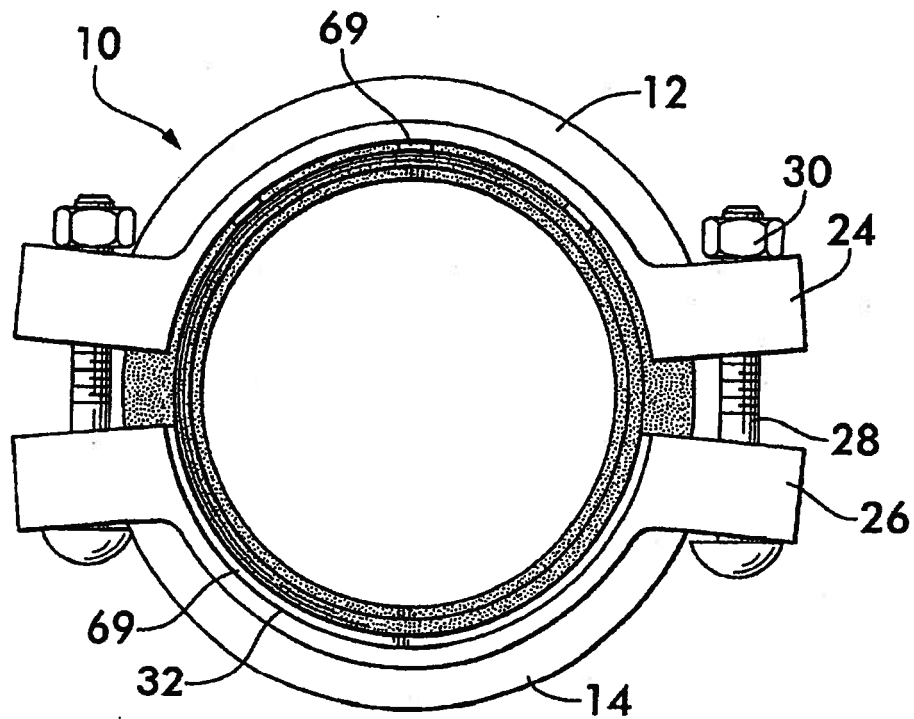
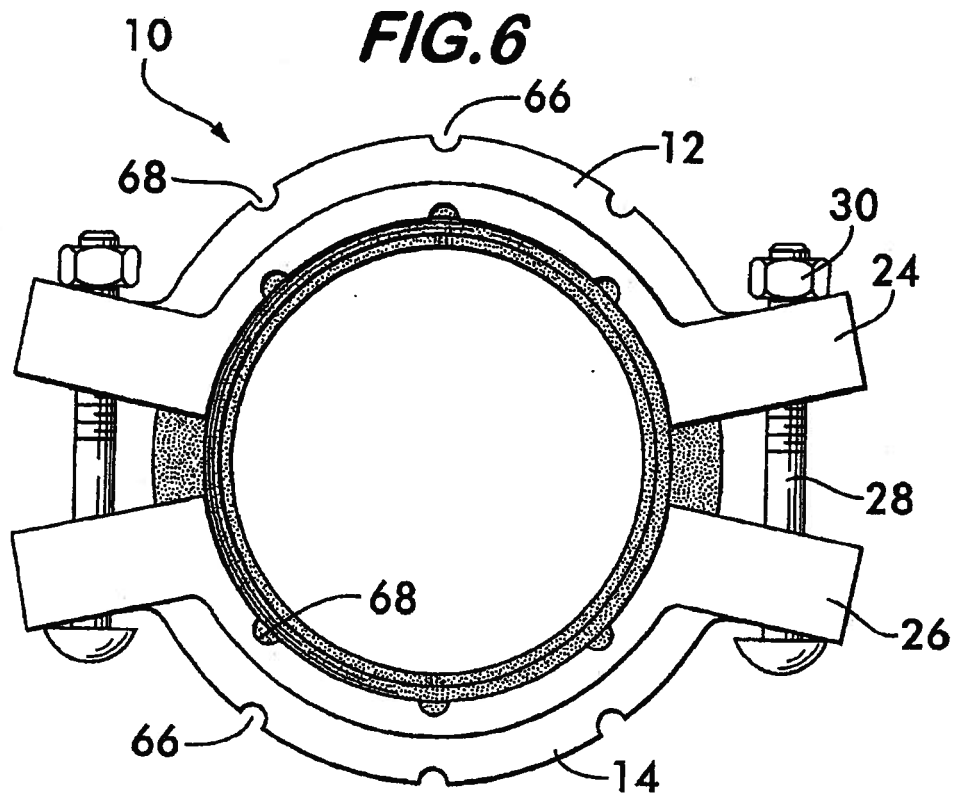


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**FIG. 7**

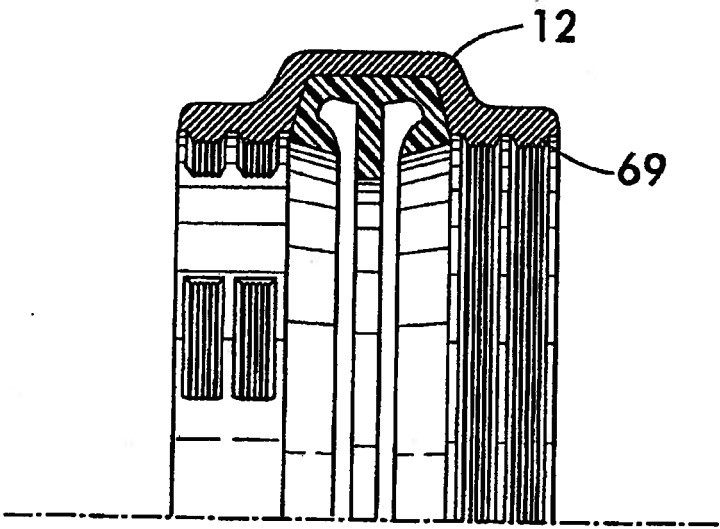
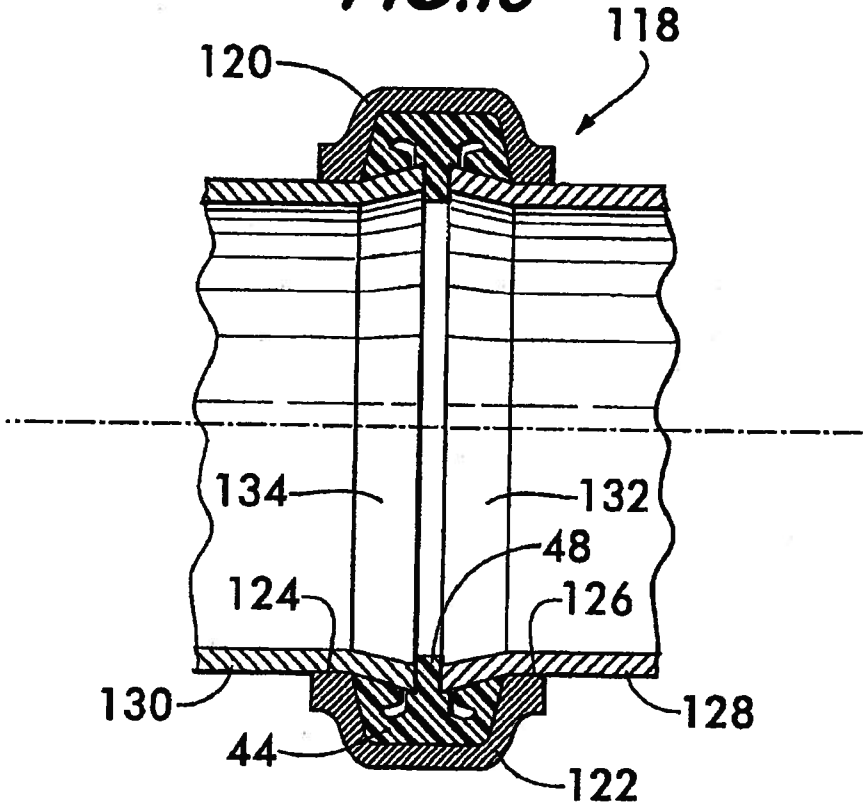
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**FIG. 13**



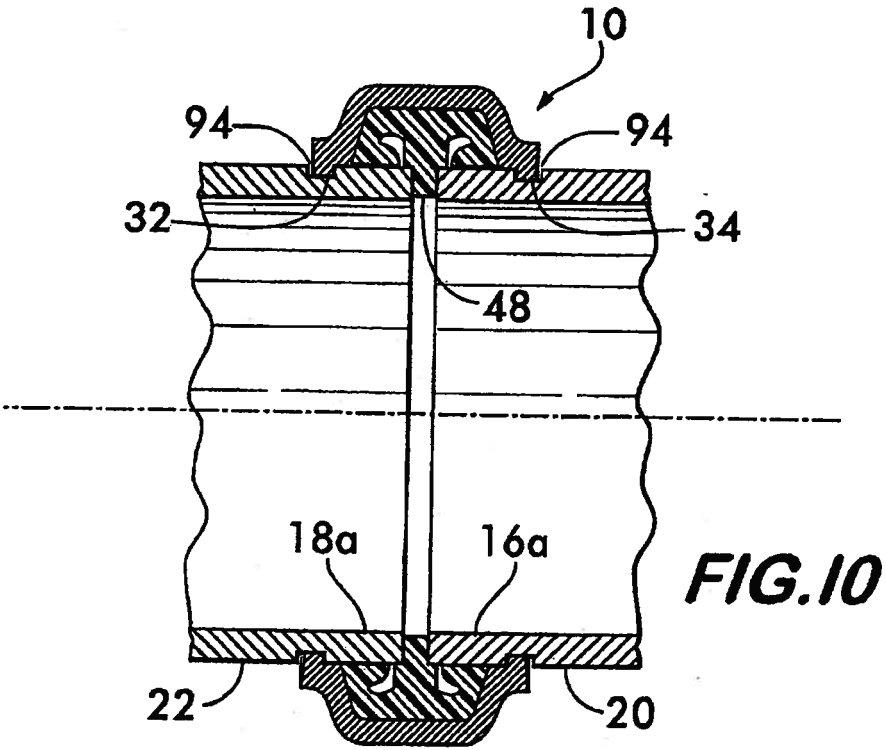
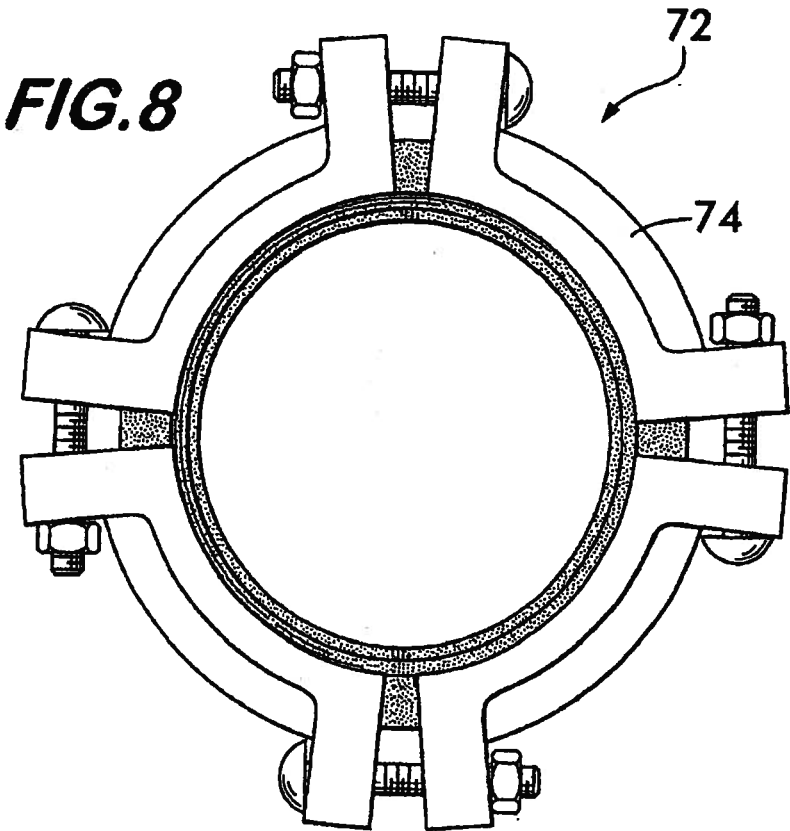
**FIG. 7A**

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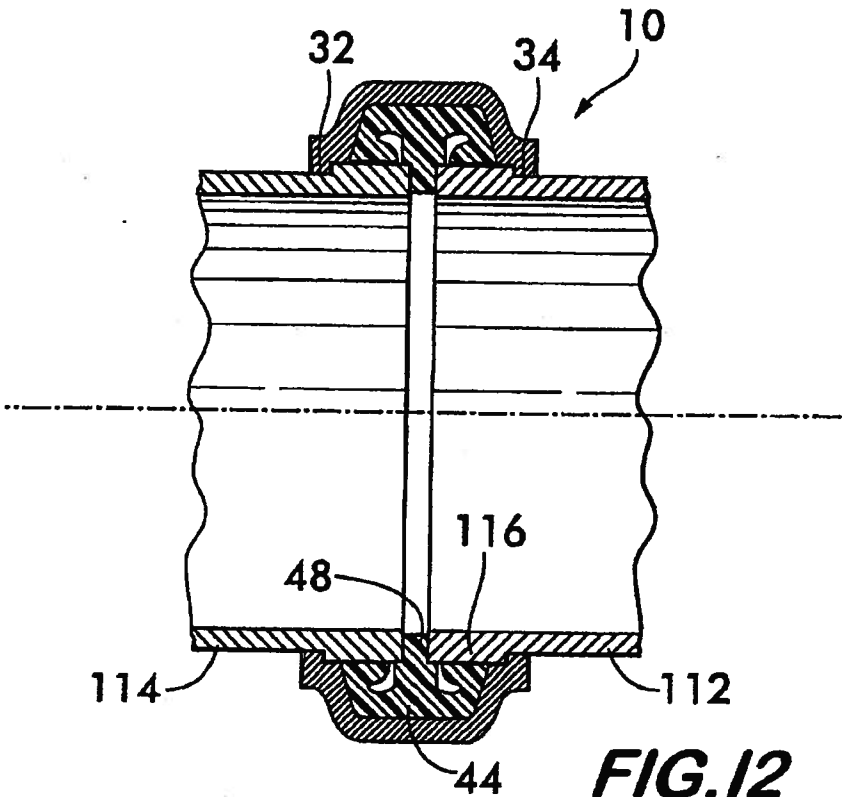
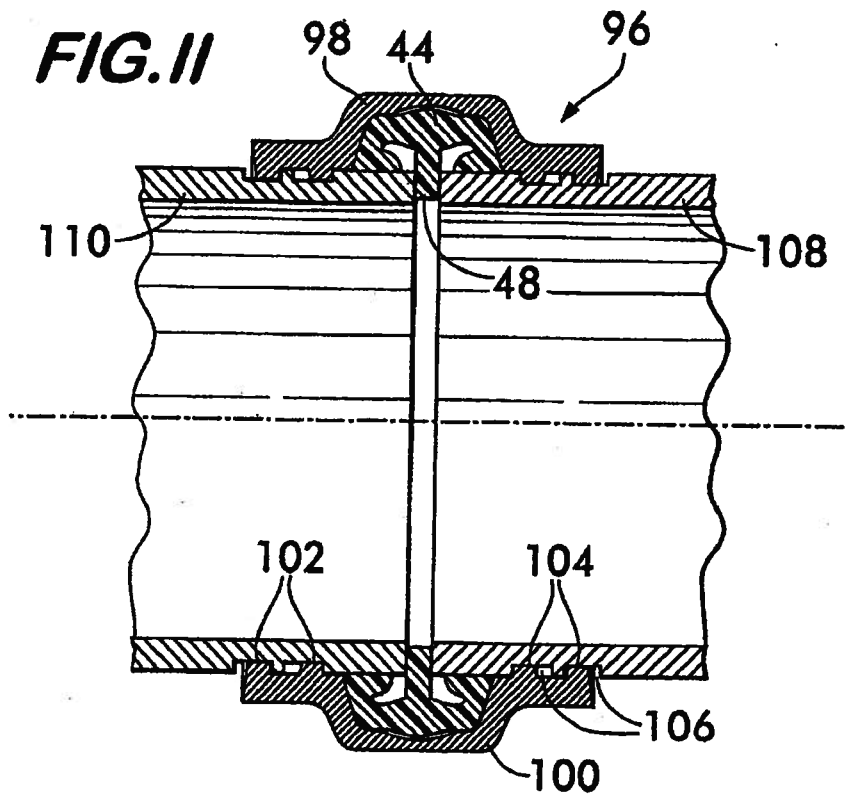
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**FIG. 11**



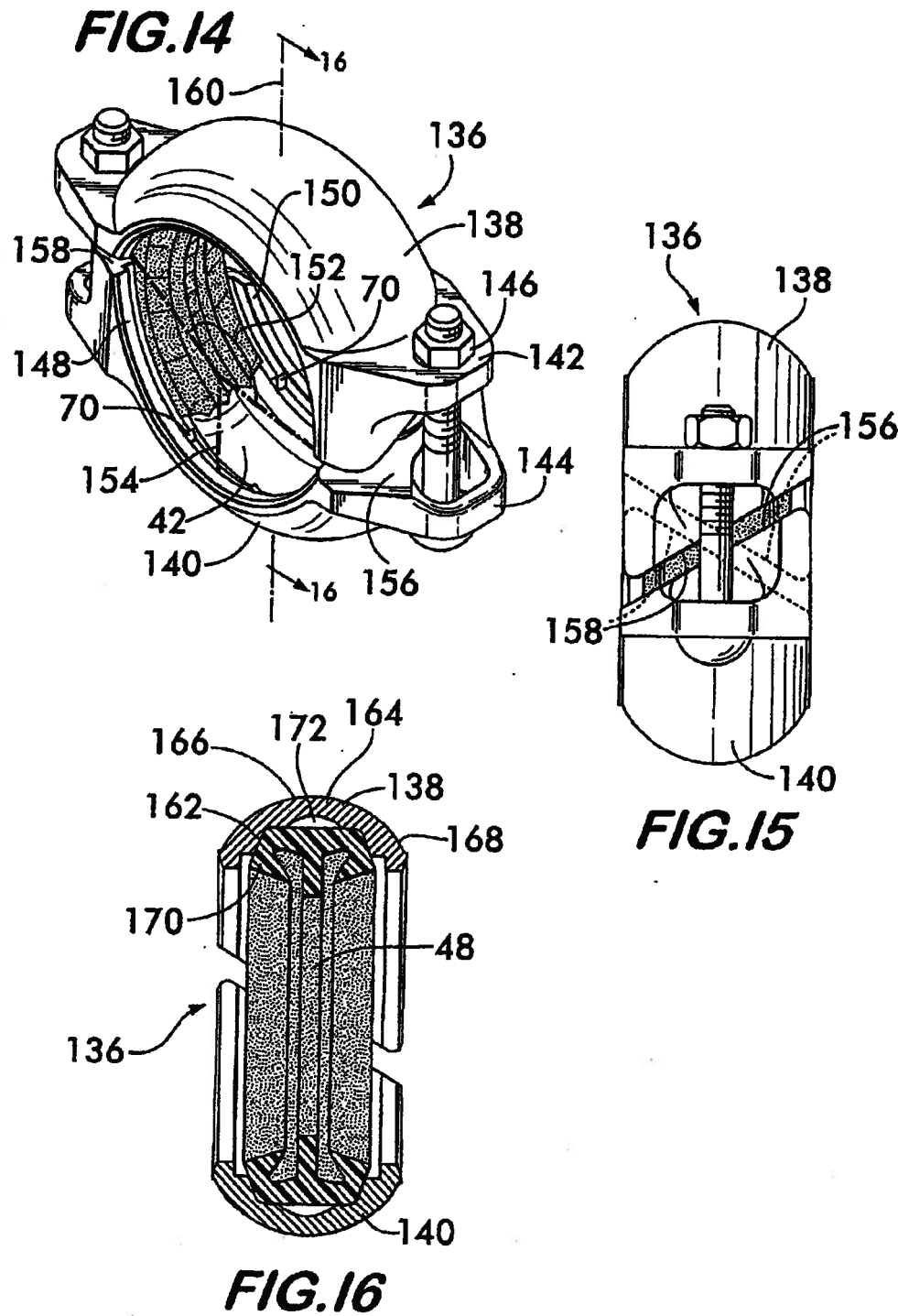
**FIG. 12**

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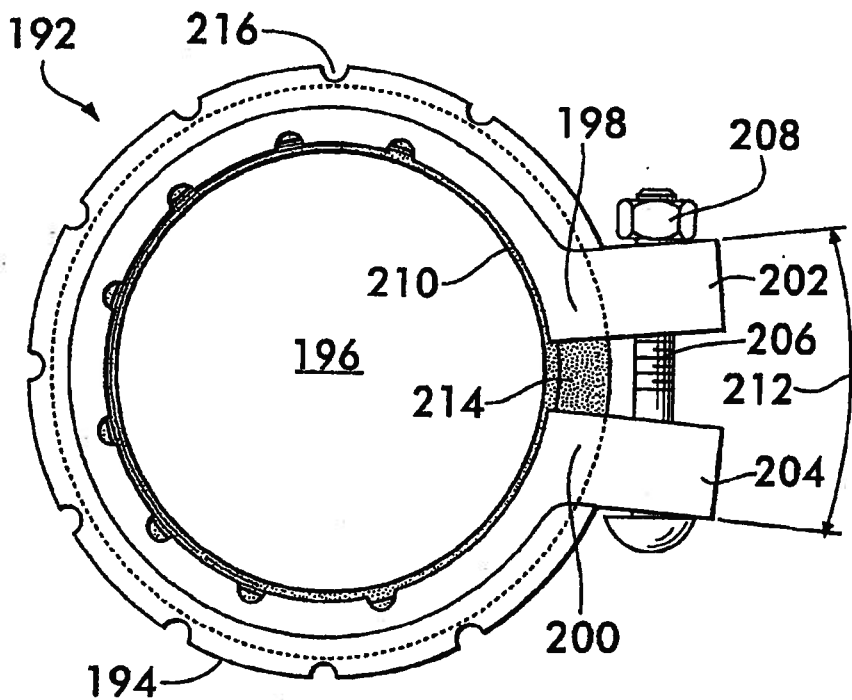
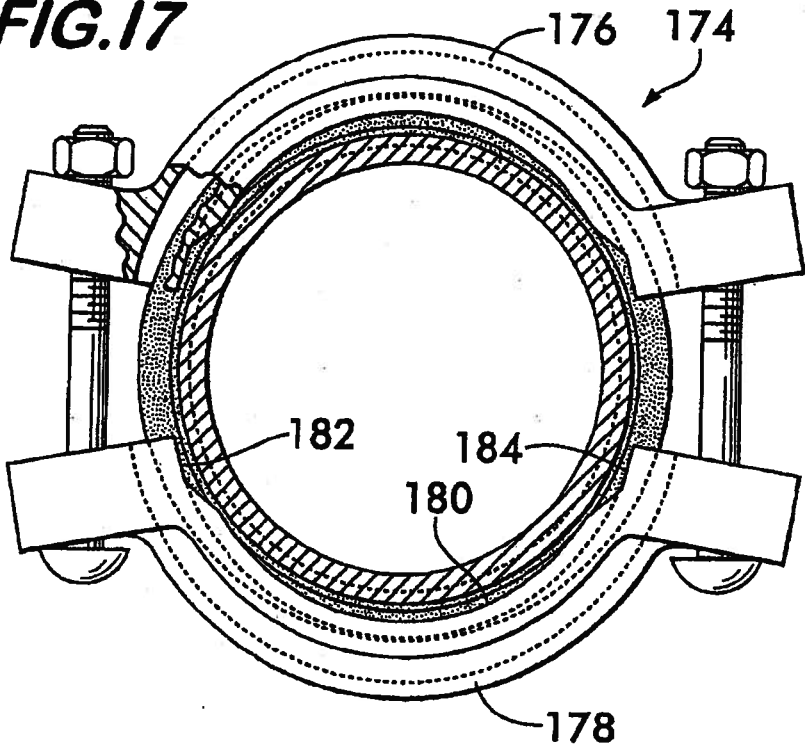
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**FIG. 17**



**FIG. 18**

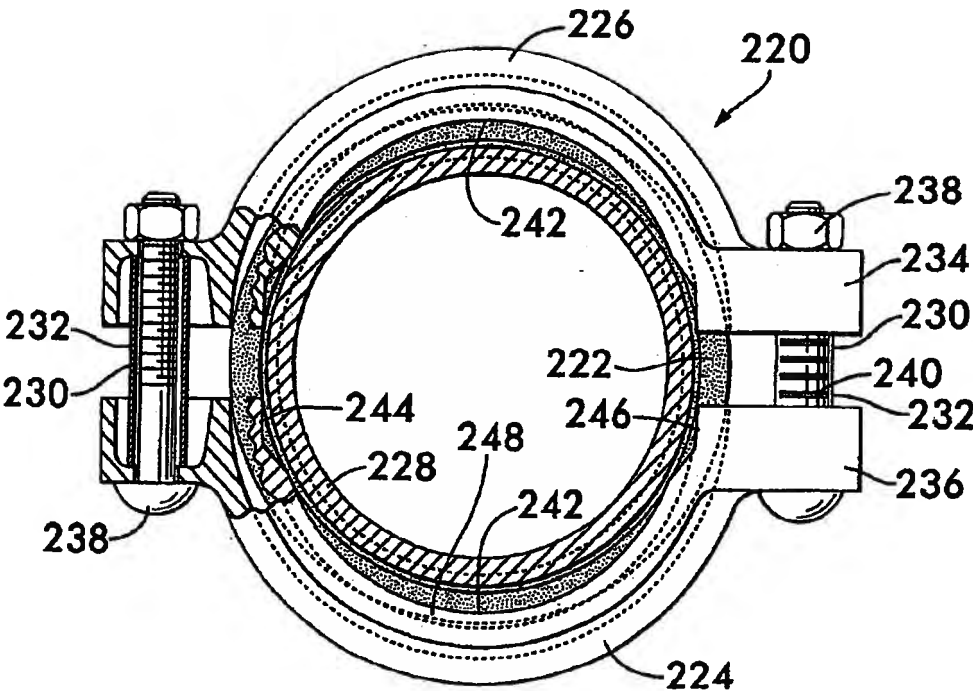
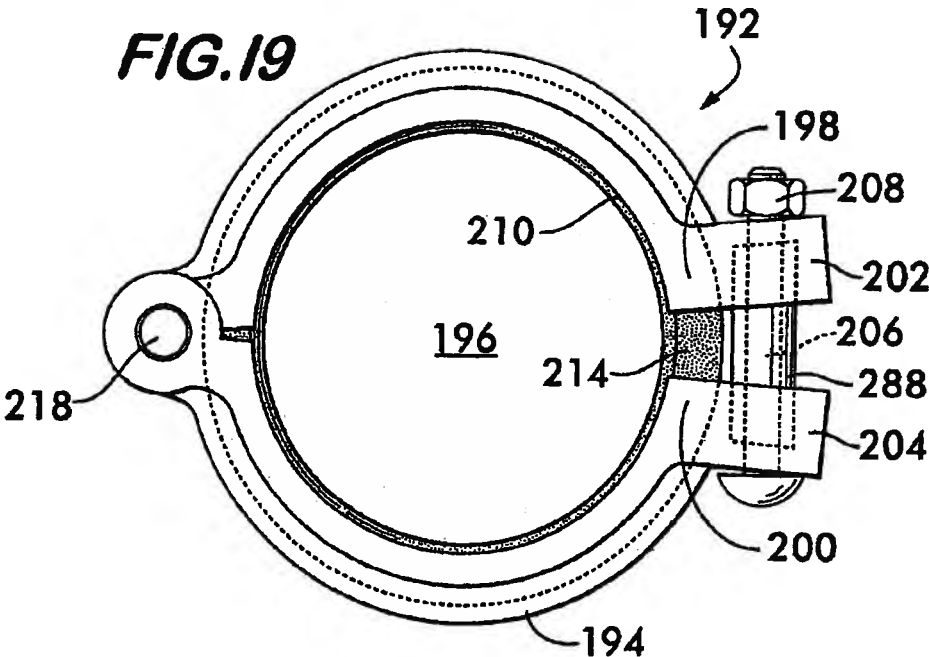
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**FIG. 19**



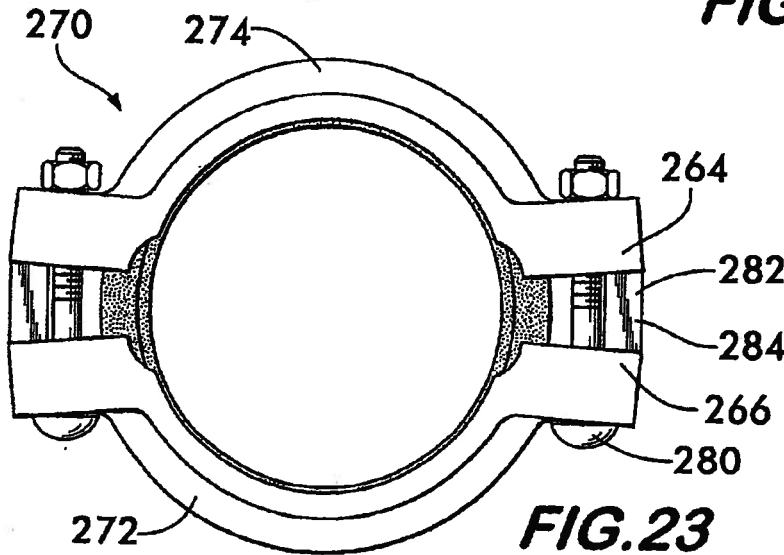
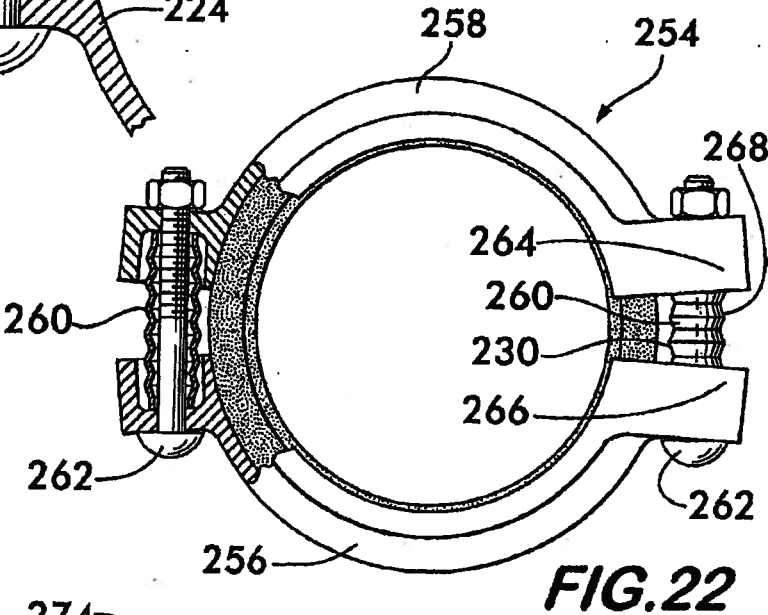
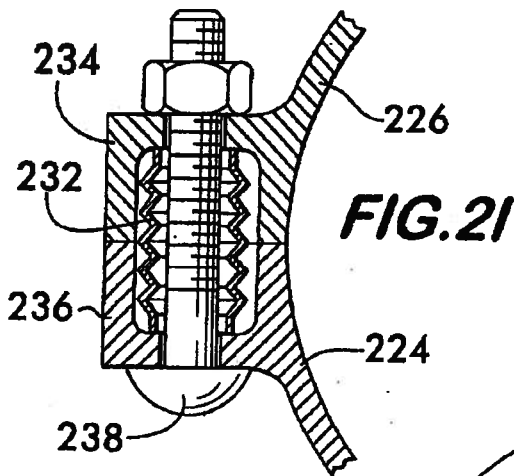
**FIG. 20**

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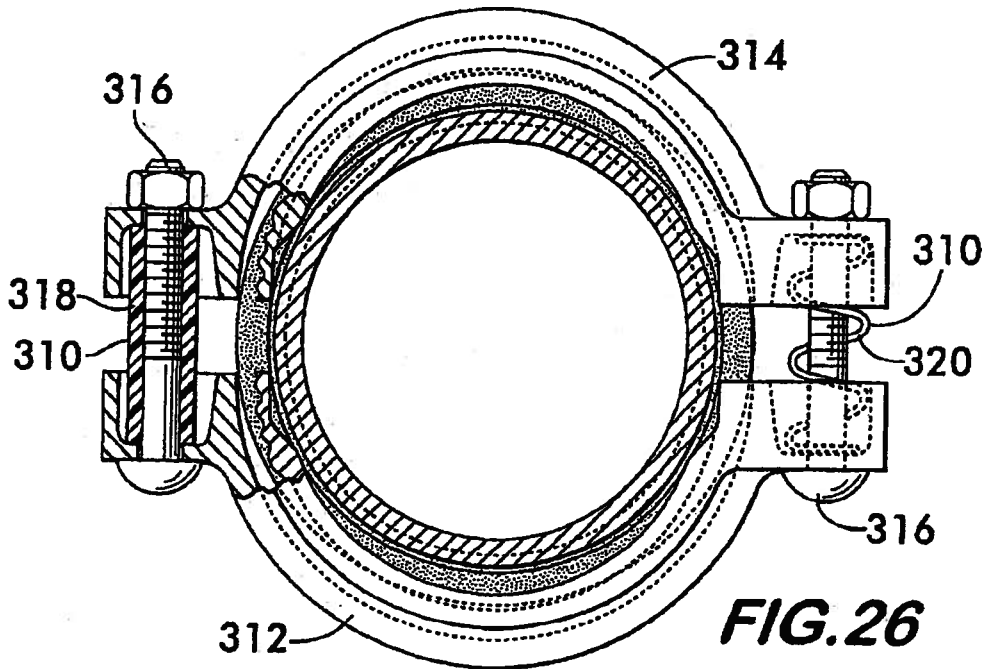
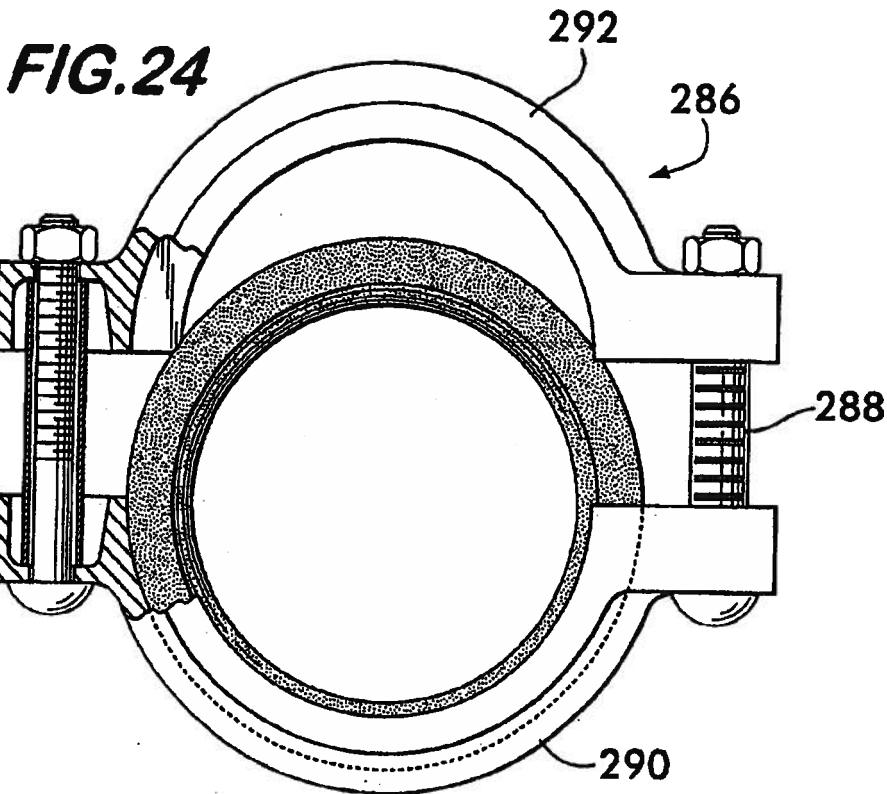


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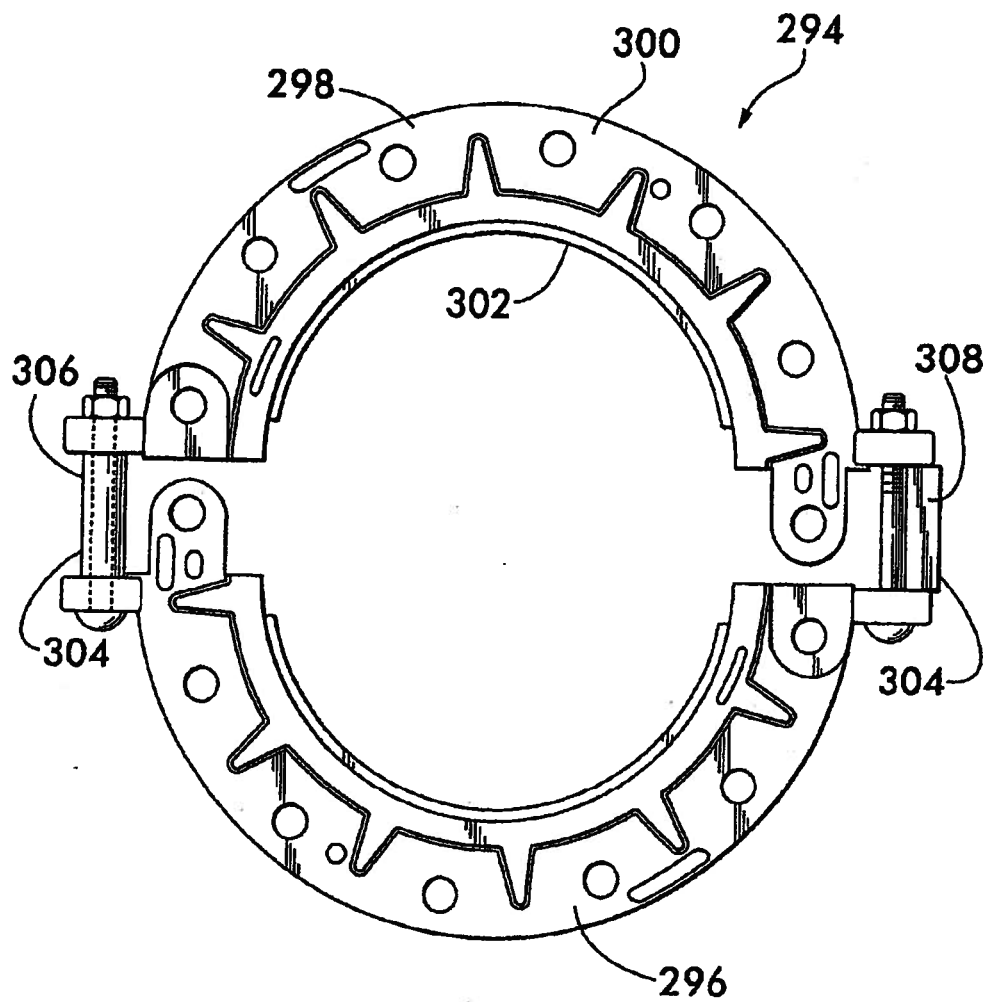


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**FIG. 25**



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## DEFORMABLE MECHANICAL PIPE COUPLING

### CROSS REFERENCE TO RELATED APPLICATION

This application is a divisional of U.S. application Ser. No. 11/124,781, filed May 9, 2005, now U.S. Pat. No. 7,086,131, issued Aug. 8, 2006, which is based on and claims priority to U.S. Provisional Application No. 60/571,596, filed May 14, 2004.

### FIELD OF THE INVENTION

This invention concerns mechanical pipe couplings that are deformable to conform to pipe elements, allowing the couplings to be pre-assembled and installed as a unit.

### BACKGROUND OF THE INVENTION

Mechanical couplings for joining pipe elements together end-to-end comprise interconnectable segments that are positionable circumferentially surrounding the end portions of co-axially aligned pipe elements. The term "pipe element" is used herein to describe any pipe-like item or component having a pipe like form. Pipe elements include pipe stock, pipe fittings such as elbows, caps and tees as well as fluid control components such as valves, reducers, strainers, restrictors, pressure regulators and the like.

Each mechanical coupling segment comprises a housing having arcuate surfaces which project radially inwardly from the housing and engage plain end pipe elements or circumferential grooves that extend around each of the pipe elements to be joined. Engagement between the arcuate surfaces and the pipe elements provides mechanical restraint to the joint and ensures that the pipe elements remain coupled even under high internal pressure and external forces. The housings define an annular channel that receives a gasket or seal, typically an elastomeric ring which engages the ends of each pipe element and cooperates with the segments to provide a fluid tight seal. The segments have connection members, typically in the form of lugs which project outwardly from the housings. The lugs are adapted to receive fasteners, such as nuts and bolts, which are adjustably tightenable to draw the segments toward one another.

To ensure a good fit between the couplings and the pipe elements, the arcuate surfaces on prior art couplings have a radius of curvature that is substantially matched to the radius of curvature of the outer surface of the pipe element that it is intended to engage. For couplings used with grooved pipe elements, the radii of curvature of the arcuate surfaces are smaller than the radii of curvature of the outer surfaces of the pipe elements outside of the grooves so that the arcuate surfaces fit within and engage the grooves properly.

This geometrical relation between the arcuate surfaces of the couplings and the outer surfaces of the pipe elements in prior art couplings results in a tedious and time consuming installation process when mechanical couplings are used. Typically, the coupling is received by the technician with the segments bolted together and the ring seal captured within the segments' channels. The technician first disassembles the coupling by unbolting it, removes the ring seal, lubricates it (if not pre-lubricated) and places it around the ends of the pipe elements to be joined. Installation of the ring seal requires that it be lubricated and stretched to accommodate the pipe elements, an often difficult and messy task, as the ring seal is usually stiff and the lubrication makes manual manipulation

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of the seal difficult. With the ring seal in place on both pipe elements, the segments are then placed one at a time straddling the ends of the pipe elements and capturing the ring seal against them. During placement, the segments engage the seal, the arcuate surfaces are aligned with the grooves, the bolts are inserted through the lugs, the nuts are threaded onto the bolts and tightened, drawing the coupling segments toward one another, compressing the seal and engaging the arcuate surface within the grooves.

As evident from the previous description, installation of mechanical pipe couplings according to the prior art requires that the technician typically handle at least seven individual piece parts (and more when the coupling has more than two segments), and must totally disassemble and reassemble the coupling. Significant time, effort and expense would be saved if the technician could install a mechanical pipe coupling without first totally disassembling it and then reassembling it, piece by piece.

### SUMMARY OF THE INVENTION

The invention concerns interconnectable pipe coupling segments. Each segment is positionable straddling facing end portions of a pair of pipe elements for securing the pipe elements together in end-to-end relationship. The end portions of the pipe elements have an outer surface of substantially cylindrical profile. Each segment comprises a pair of arcuate surfaces adapted to interface with the outer surfaces of the pipe elements. The arcuate surfaces are in spaced apart relation to one another. The arcuate surfaces subtend an angle of less than 180° and having a radius of curvature greater than the radius of curvature of the outer surfaces of the pipe elements. Each segment further includes connection members for adjustably connecting one coupling segment to another. The connection members are adjustably tightenable for drawing the arcuate surfaces of the segments together. The segments are deformable upon adjustable tightening of the connection members so as to conform the curvature of the arcuate surfaces to the outer surfaces of the pipe elements.

Preferably, the segments are substantially elastically deformable, and the arcuate surfaces project radially inwardly from the segments. The connection members comprise a pair of projections extending outwardly from the ends of each of the segments. The projections are adapted to receive fasteners for adjustably connecting the segments to one another, the fasteners being adjustably tightenable for drawing the arcuate surfaces of the segments together into engagement with the outer surfaces of the pipe elements. Preferably, the projections comprise lugs having an aperture therethrough adapted to receive the fastener.

The invention also concerns a pipe coupling positionable straddling facing end portions of a pair of pipe elements for securing the pipe elements together in end-to-end relationship. Again, the end portions of the pipe elements have an outer surface of substantially cylindrical profile. The pipe coupling comprises first and second coupling segments. Each coupling segment has a pair of arcuate surfaces adapted to interface with the outer surfaces of the pipe elements. The arcuate surfaces are in spaced apart relation to one another. The arcuate surfaces subtend an angle of less than 180° and have a radius of curvature greater than the radius of curvature of the outer surfaces of the pipe elements. Each coupling segment has connection members for adjustably connecting one coupling segment to another. The connection members are adjustably tightenable for drawing the arcuate surfaces of the segments together. The segments are deformable upon adjustable tightening of the connection members so as to

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substantially conform the curvature of the arcuate surfaces to the outer surfaces of the pipe elements.

A pipe coupling also includes a flexible, resilient seal. The seal is preferably a substantially circular ring having an inner diameter sized to receive the pipe elements. The seal is positioned between the arcuate surfaces of the first and second coupling segments. The seal has an outer diameter sized to position the first and second coupling segments in spaced apart relation from one another far enough to thereby allow the pipe elements to be inserted between the first and second coupling segments into end-to-end relationship. The seal is deformable when the connection members are adjustably tightened to draw the arcuate surfaces together and conform the curvature of the segments to the outer surfaces of the pipe elements.

Preferably, each of the first and second segments further comprises a pair of angularly oriented surfaces positioned adjacent to each of the connection members. The angularly oriented surfaces on each segment have opposite slopes. The angularly oriented surfaces on one segment are in facing relation with the angularly oriented surfaces on the other segment. The angularly oriented segments engage one another when the segments are drawn together and cause the segments to rotate relatively to one another about an axis substantially perpendicular to the pipe elements. The drawing together and rotation of the segments forces engagement between the arcuate surfaces and the grooves to provide rigidity about all axes of the joint.

In another embodiment, a pipe coupling comprises an arcuate band having first and second ends in substantially facing relation. The ends are spaced apart from one another. The band surrounds and defines a central space. First and second arcuate surfaces are mounted lengthwise along the band on a side thereof. The arcuate surfaces are in spaced relation alongside one another and project substantially radially inwardly into the central space. The arcuate surfaces have a radius of curvature greater than the radius of curvature of the outer surfaces of the pipe elements. The end portions of the pipe elements are insertable into the central space. Connection members are mounted on the first and second ends of the segments. The connection members are adjustably tightenable for drawing the first and second ends toward one another. The band is deformable, allowing the first and second ends to move toward one another upon adjustable tightening of the connection members. The arcuate surfaces are thereby brought into engagement with the outer surfaces of the pipe elements, their curvature substantially conforming to the curvature of the outer surfaces of the pipe elements.

Deformation of the band may be elastic, plastic, or may be facilitated by a hinge positioned between the first and second ends. The hinge allows a first portion of the band to pivot relatively to a second portion of the band for receiving the pipes within the central space.

The invention also includes a method of securing facing end portions of pipe elements together in end-to-end relationship. The method comprises the steps of:

(A) providing a pipe coupling having a plurality of coupling segments attached to one another end-to-end surrounding a central space, the coupling segments having arcuate surfaces adapted to interface with the outer surfaces of the pipes;

(B) inserting the end portions of the pipe elements into the central space; and

(C) deforming the coupling segments so as to conform the curvature of the arcuate surfaces of the coupling segments to the outer surfaces of the pipe elements.

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#### BRIEF DESCRIPTION OF THE DRAWINGS

FIGS. 1-1B are longitudinal cross-sectional views of a deformable mechanical pipe coupling according to the invention;

FIGS. 2 and 3 are partial cross-sectional views of the pipe coupling shown in FIG. 1;

FIGS. 4 and 5 are perspective views, partially cut away, of seals used with pipe couplings according to the invention;

FIGS. 6-7 and 8 are axial views of various pipe coupling embodiments according to the invention;

FIGS. 7A and 9-13 are longitudinal sectional views of pipe coupling embodiments according to the invention;

FIG. 14 is a perspective view of a pipe coupling according to the invention;

FIG. 15 is a side view of the pipe coupling shown in FIG. 14;

FIG. 16 is a cross-sectional view taken at line 16-16 in FIG. 14;

FIG. 17 is an axial view, partially cut away, of pipe coupling embodiment according to the invention;

FIG. 18 is an axial view of a pipe coupling embodiment according to the invention;

FIG. 19 is an axial view of a pipe coupling embodiment according to the invention;

FIG. 20 is an axial view, partially cut away, of a pipe coupling embodiment according to the invention;

FIG. 21 is a partial sectional view of the pipe coupling shown in FIG. 20;

FIG. 22 is an axial view, partially cut away, of a pipe coupling embodiment according to the invention;

FIG. 23 is an axial view of a pipe coupling embodiment according to the invention; and

FIG. 24-26 are axial views of a pipe coupling embodiments according to the invention.

#### DETAILED DESCRIPTION OF THE EMBODIMENTS

FIGS. 1 and 2 show a pipe coupling 10 according to the invention. Coupling 10 is formed from coupling segments 12 and 14 which are interconnectable with one another to straddle end portions 16a and 18a of pipe elements 16 and 18 to secure the pipe elements together in end-to-end relationship. The end portions of the pipe elements have respective outer surfaces 20 and 22 of substantially cylindrical profile.

Interconnection of the coupling segments 12 and 14 is effected by connection members, preferably in the form of lugs 24 and 26 best shown in FIG. 2. The lugs are preferably positioned at each end of each segment and project outwardly from the segments. Lugs 24 and 26 are positioned in facing relation to one another and adapted to receive fasteners, preferably in the form of bolts 28 and nuts 30 which are adjustably tightenable and cooperate with the lugs 24 and 26 for adjustably connecting the coupling segments to one another as discussed in further detail below.

As best shown in FIG. 1, each segment 12 and 14 comprises a pair of arcuate surfaces 32 and 34. The arcuate surfaces are in spaced apart relation to one another and preferably project radially inwardly toward the pipe elements 16 and 18. The surfaces extend from a housing 36 having sidewalls 38 joined to a backwall 40, the sidewalls and backwall forming a channel 42 that receives a seal 44.

Examples of seals 44 are shown in FIGS. 4 and 5. Seal 44 is preferably a flexible, resilient ring formed from elastomeric material. The seal may have lips 46 that use the internal pressure within the pipes to increase the sealing force

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between the seal and the outer surfaces 20 and 22 of the pipe elements 16 and 18. As shown in FIG. 5, seal 44 may also have a tongue 48 positioned between the lips 46, the tongue extending circumferentially around the seal and projecting radially inwardly. Tongue 48 provides a stop surface that engages the ends of pipe elements 16 and 18 to ensure proper positioning of the seal 44 relatively to the pipe elements as described in detail below. Engagement of the pipe elements with tongue 48 also effects alignment of the arcuate surfaces with the grooves (if present), or with alignment marks on the outside surface of the pipe elements.

As illustrated in FIG. 2, arcuate surfaces 32 and 34 have radii of curvature 50 greater than the radii of curvature 52 of the outer surfaces 20 and 22 of pipe elements 16 and 18. Furthermore, the arcuate surfaces 32 subtend an angle 54 of less than 180°. Angles 54 between about 40° and about 179° are practical. As a result of this arcuate surface geometry, segments 12 and 14 may be pre-assembled separated from one another such that pipe elements 16 and 18 may be inserted directly into the coupling 10 as shown in FIG. 1 without first disassembling the coupling. This feature provides a significant advantage over prior art couplings which must be assembled onto the pipe ends piece by piece. Joining of the pipe ends with a coupling 10 according to the invention proceeds much more smoothly and quickly than with prior art couplings because the technician handles fewer pieces and does not have to thread nuts onto bolts. In the embodiment shown in FIG. 1, the seal 44 has an outer diameter 56 sized to hold the coupling segments 12 and 14 in spaced apart relation sufficient to allow the pipe ends to be inserted as described above. The seal inner diameter 58 is sized to receive the end portions 16a and 18a of the pipe elements simply by pushing the coupling over the pipe element or by inserting the pipe elements into the coupling. Other embodiments having different features for supporting the segments in spaced relation are described below.

After both pipe elements 16 and 18 are inserted into coupling 10 as shown in FIG. 1A, nuts 30 are tightened (see also FIG. 2). The nuts 30 cooperate with their bolts 28 to draw the arcuate surfaces 32 and 34 on segment 12 toward those on segment 14. Tightening of the nuts exerts a force on the lugs 24 and 26 which brings the segments into contact with the pipe elements and causes the segments 12 and 14 to deform such that the radius of curvature 50 of the arcuate surfaces 32 and 34 substantially conforms to the radius of curvature 52 of the pipe elements 16 and 18. This action is illustrated by comparing FIGS. 2 and 3 and 1A and 1B, wherein the gap 60 between the arcuate surfaces and the pipe outer surfaces diminishes as the arcuate surfaces are brought into engagement with the outer surfaces of the pipe ends. Deformation of the segments 12 and 14 is preferably substantially elastic, allowing the segments to spring back substantially to their original shape when the nuts 30 are loosened, thereby permitting the coupling 10 to be reused in the manner according to the invention as described herein. The segments may also be designed to have significant plastic deformation, wherein the deformation imparts a permanent set to the segments. For practical couplings, there will generally be some degree of both plastic and elastic deformation occurring in the segments as a result of tightening the fasteners. Additionally, when the segments 12 and 14 are in the undeformed state (FIG. 2), the lugs 24 and 26 may be angularly oriented in relation to one another. Relative angles 62 up to about 10° are practical. As shown in FIG. 3, the relative angular orientation of the lugs 24 and 26 is reduced as the segments are deformed, and the geometry may be designed such that the lugs are substantially parallel once the arcuate surfaces 32 and 34

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substantially conform to the outer surfaces 20 and 22. This is preferred because, when fully tightened, the bolt head and nut will be in substantially flat contact with the lugs, thereby avoiding induced bending moments in the bolts which can cause permanent deformation of the bolts. The seal 44 is also deformed by this process, as shown in FIG. 1B, with the lips 46 coming into full engagement with the pipe element outer surfaces 20 and 22. Because the seal 44 is substantially incompressible, it must be provided with space into which it may expand when compressed by the segments. This space is provided by a concavity 64 positioned in the backwall 40 between the sidewalls 38. Concavity 64 may take virtually any practical shape and allows for volume change of the seal when it is heated or exposed to fluids thereby distributing the deformation of the seal more evenly over its circumference and mitigating the tendency of the seal to extrude outwardly from between the segments between the lugs. The concavity also prevents tongue 48, if present, from being forced between the ends of the pipe elements and impede flow there-through.

As shown in FIGS. 2 and 3, for the preassembled coupling 10, it is advantageous to hold nuts 30 in a position on bolts 28 that will maintain the segments 12 and 14 in the desired spaced apart relation as determined by contact between the segments and the seal 44. This is conveniently accomplished by deforming the threads 29 of bolts 28, preferably by staking. Staking the bolts hinders the rotation of the nuts and prevents them from unscrewing from the bolts under the effect of vibration, for example, during shipping, and keeps the coupling in the preassembled state with all of its parts together prior to installation. The staking is readily overcome when the nuts are tightened with a wrench.

The bending stiffness of the segments may be tuned to control the amount of force necessary to deform them in order to reduce the required assembly torque and mitigate galling between the nut and the lug. As shown in FIG. 6, sections of increased bending flexibility 66 may be formed in the housing 36 of the segments 12 and 14 by reducing the area moment of inertia of the segment. This reduction is preferably achieved by adding one or more cut-outs 68 in either or both the backwall 40 and the arcuate surfaces 32 and 34.

Alternately, as shown in FIG. 7, the segments may have arcuate surfaces 32 and 34 (not shown) comprising inwardly projecting teeth 69. Teeth 69 engage outer surfaces of the pipe elements to provide mechanical restraint, and are especially advantageous when used with plain end pipe elements. Teeth 69 may be substantially continuous, as shown on segment 14, or intermittent, as shown on segment 12. Single teeth, preferable for small couplings, are also feasible. As shown in FIG. 7A, teeth 69 may also be arranged in pairs on opposite sides of the segment to increase the mechanical restraint provided by the coupling.

Although couplings according to the invention are described above as comprised of two segments, this is by way of example only. Couplings with more than two segments are feasible and preferred for larger diameter pipes due to the manufacturing costs, as reducing the size of the segments is economically advantageous. A further advantage is that the spacing between the lugs is reduced, requiring fewer turns of the nut and shorter bolts. Standard depth sockets may thereby be used during installation. FIG. 8 shows an example of a coupling embodiment 72 having four segments 74 similar to those described above.

Couplings have thus far been shown wherein all of the arcuate surfaces have substantially the same radius of curvature. Whereas such a configuration is appropriate for joining pipes having substantially the same diameter to one another,



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FIG. 9 shows a coupling embodiment 76 for coupling pipe elements of different diameters. Coupling 76 is formed of two segments 78 and 80 (although it may have more than two segments). Each segment has a first arcuate surface 82 having a first radius of curvature 84, and a second arcuate surface 86 having a second radius of curvature 88 smaller than the first radius of curvature 84. This allows coupling 76 to join a pipe element 90 having a larger diameter to a pipe element 92 having a smaller diameter. Analogous to the couplings described above, the radius of curvature 84 is greater than the radius of curvature of the outer surface of pipe element 90, and the radius of curvature 88 is greater than the radius of curvature of the pipe element 92. This geometric relationship allows the pipe elements 90 and 92 to be inserted into a pre-assembled coupling 76 and achieve the advantages of the invention. The coupling segments 78 and 80 deform upon the application of force by adjustable connection members to conform the radii of curvature to the outer surface of the pipe elements.

In a preferred embodiment, shown in FIG. 10, the inwardly projecting arcuate surfaces 32 and 34 of coupling 10 engage grooves 94 formed in the outer surfaces 20 and 22 of pipe element end portions 16a and 18a. Interaction between the arcuate surfaces 32 and 34 with their respective grooves 94 permits the coupling to provide relatively high end restraint to withstand forces caused by internal pressure or external loads. To obtain higher end restraint, it is found useful to add a second set of arcuate surfaces that interact with a second set of grooves in the pipe elements. This embodiment is illustrated in FIG. 11, wherein a coupling 96 is comprised of segments 98 and 100, each segment having two pairs of arcuate surfaces 102 and 104 that project inwardly from the segments. The arcuate surface pairs are in substantially parallel, spaced relation to one another and engage pairs of grooves 106 in the surfaces of the pipe elements 108 and 110 which they connect together.

In another embodiment, shown in FIG. 12, couplings according to the invention such as 10 may be used with pipe elements 112 and 114 having raised circumferential shoulders 116 that are engaged by the arcuate surfaces 32 and 34 of the segments 12 and 14. Alternately, as shown in FIG. 13, a coupling 118 according to the invention having segments 120 and 122 with respective arcuate surfaces 124 and 126 is used with pipe elements 128 and 130 having flared end portions 132 and 134. Note that in the example embodiments shown in FIGS. 9-13, the seal 44 has the tongue 48 which is effectively used to position the pipe ends within the coupling upon insertion, the tongue acting as a pipe stop to aid in locating the pipe ends at the proper depth within the couplings.

Another coupling embodiment 136 is shown in FIG. 14. Coupling 136 is comprised of two segments 138 and 140 from which lugs 142 and 144 extend, the lugs cooperating with fasteners 146 to act as connection members for adjustably connecting one coupling segment to another. As described above, each segment has a pair of arcuate surfaces 148, 150, each preferably projecting radially inwardly from the segments. The arcuate surfaces subtend an angle 152 less than 180° and have a radius of curvature 154 greater than the radius of curvature of the pipe elements which the coupling is to join together. Anti-rotation teeth 70 are positioned adjacent to the arcuate surfaces and project radially inwardly to engage the pipe elements and provide torsional rigidity.

As best shown in FIG. 15, each segment 138 and 140 has a pair of angularly oriented surface portions 156 and 158 located adjacent to each of the lugs 142 and 144. As illustrated, the slope of surface portion 156 may be opposite to the slope of surface portion 158 on each segment. (Both surfaces

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could also be sloped in the same direction as well.) This opposite slope relationship between the surfaces on a segment results in surfaces having compatible slopes being positioned in facing relation in a pre-assembled coupling as shown in FIG. 15. When the fasteners 146 are tightened, conforming the arcuate surfaces to the outer surfaces of the pipe elements, the angular surface portions 156 and 158 on each segment engage and slide relatively to one another, causing the segments to draw together and rotate relatively to one another in opposite directions about an axis 160 oriented substantially perpendicularly to the axis of the pipe elements being joined. These motions of the segments 138 and 140 causes the arcuate surfaces 148 and 150 to engage grooves in the pipe elements and adds rigidity to all axes of the joint as previously described. For coupling segments having surface portions with the same slopes, the couplings move along the pipe in opposite directions relatively to one another with similar effect.

As shown in cross section in FIG. 16, the segments 138 and 140 forming the coupling 136 have a channel 162 defined by a housing 164. The housing is formed from a back wall 166 and sidewalls 168, and receives a seal 170 which is sized to position the segments 138 and 140 in spaced apart relation so as to allow insertion of pipe elements into the pre-assembled coupling shown in FIG. 14. A concavity 172 is provided in the back wall to provide a space for volume change of the seal when it is heated or exposed to fluids as well as to prevent tongue 48 from being forced between the ends of the pipe elements and impede flow therethrough due to compression of the seal.

In another coupling embodiment, shown in FIG. 17, the coupling 174 again is comprised of at least two coupling segments 176 and 178, each having inwardly projecting arcuate surfaces 180 as described above. However, arcuate surfaces 180 have notches 182 and 184 positioned at opposite ends. The notches 182 and 184 provide clearance at the 3 o'clock and 9 o'clock positions of the coupling where it is most needed to permit pipe elements to be inserted into the pre-assembled coupling 174. The availability of increased clearance at these locations allows the coupling segments 176 and 178 to be spaced closer to one another in the pre-assembled configuration than would be the case if the clearance was not available at the ends of the surfaces. By having the segments of the preassembled coupling closer together, the amount of deformation required to conform the arcuate surfaces to the pipe element outer surface is reduced and thereby the energy required to tighten the fasteners.

Another coupling embodiment 192 according to the invention is shown in FIG. 18. Coupling 192 comprises an arcuate band 194 surrounding a central space 196. Band 194 has opposite ends 198 and 200 positioned in facing relation to one another. Ends 198 and 200 are in spaced relation in the pre-assembled coupling and have connection members mounted thereon, preferably in the form of projecting lugs 202 and 204 adapted to receive a fastener such as bolt 206 and nut 208. The bolt and nut cooperate with the lugs to deform the band 194 and bring the ends 198 and 200 toward one another after pipe elements have been inserted into the central space 196 for coupling in end-to-end relationship. Band 194 has a pair of arcuate surfaces 210, only one of which is visible in the figure. The arcuate surfaces are in spaced relation lengthwise of one another as illustrated in FIG. 10 and described above for other embodiments. The arcuate surfaces 210 have a greater radius of curvature than the outer surface of the pipe ends that the coupling is to join together. This geometric configuration, and the separation of the ends 198 and 200 allows the pipe elements to be inserted into central space 196. Upon tightening

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of the nut 208 the band 194 is deformed such that the radius of curvature of the arcuate surfaces 210 are forced to conform with the radius of curvature of the outside surface of the pipe elements which they engage. Note that in the preassembled state, projecting lugs 202 and 204 are preferably angularly oriented with respect to one another, having a relative angle 212 up to about 20°. Tightening of the fastener draws the lugs toward each other, and results in decreasing the relative angle 212, preferably to the point wherein the lugs are substantially parallel to one another. This is particularly advantageous for a flexible coupling which does not depend upon the pipe elements to form a reaction point to cause the deformation in combination with the bolts, the friction incurred at the reaction points inhibiting flexibility.

Coupling 192 includes a seal 214 positioned within the band 194 between the arcuate surfaces 210. Seal 214 may be similar to those illustrated in FIGS. 4 and 5 and sized to receive the pipe elements for creating a fluid tight seal when the band is deformed.

The bending flexibility of coupling 192 may be adjusted by reducing the area moment of inertia of band 194. Such adjustments may be effected by positioning cut-outs 216 in the band. Alternately, as shown in FIG. 19, a hinge 218 may be provided between the ends 198 and 200. Hinge 218 is preferably positioned equidistant from the ends of the band and provides infinite bending flexibility, reducing the torque needed on the fastener to draw the ends 198 and 200 toward one another. The band 194 will still deform as the arcuate surfaces 210 engage the outer surfaces of pipe elements to conform the radii of the surfaces with that of the pipe element outer surfaces. When the hinge is present, the seal 214 is sized so as to maintain the lugs 202 and 204 in spaced relation so that pipe elements may be inserted. For both the hinged and hingeless versions of the coupling described above, the arcuate surfaces preferably project radially inwardly from the band and may have different radii of curvature from each other, as illustrated in FIG. 9, to allow the coupling 192 to be used to join pipes having different diameters.

FIG. 20 illustrates a pre-assembled coupling 220 that does not depend on the seal 222 to maintain its segments 224 and 226 in spaced apart relation and ready to receive pipe elements such as 228. Coupling 220 has spacers 230 that extend between segments 224 and 226 and maintain the segments in spaced apart relation. In this example embodiment, the spacers 230 comprise collapsible tubes 232 that are positioned between facing lugs 234 and 236 that extend from the segments. Tubes 232 are preferably thin walled and circular in cross section and are arranged coaxially surrounding the fasteners 238. The tubes may be made of lightweight metal or a polymer material such as polypropylene and may have score lines 240 in their surface to create weakened regions that facilitate collapse of the tube under compressive loads applied by the fasteners 238. Other materials, such as cardboard and rubber are also feasible. The tubes are designed to be strong enough to support the segments in spaced relation during shipping, handling and installation, but collapse at a predetermined compressive load that a technician may apply, preferably by manually tightening the fasteners with a wrench.

In use, pipe elements to be joined end-to-end are inserted between the segments 224 and 226. Fasteners 238 are then tightened to draw the segments toward each other and into engagement with the pipe elements. Tightening of the fasteners places the tubes 232 under a compressive load, and the tubes buckle and collapse as shown in FIG. 21 when the

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predetermined load is achieved to allow the segments to move toward one another and engage the pipe elements to effect the joint.

Spacers positioned between the segments may be used with any type of mechanical coupling. Note that in FIGS. 20 and 21, the segments 224 and 226 have arcuate surfaces 242 with a radius of curvature that is substantially the same as the radius of curvature of the outer surface of pipe element 228 which they are designed to engage. To provide clearance between the pipe element 228 and the segments allowing the pipe element to be inserted into the coupling while still maintaining a reasonable fastener length, notches 244 and 246 are positioned at opposite ends of the arcuate surfaces 242 as best shown in FIG. 20. The notches provide clearance at the 3 o'clock and 9 o'clock positions of the coupling to permit pipe elements to be inserted into the pre-assembled coupling 220.

FIG. 22 illustrates another coupling embodiment 254 having spacers 230 between segments 256 and 258 comprising the coupling. In this example, the spacers 230 comprise tubes 260 again positioned coaxially with fasteners 262 and between facing lugs 264 and 266 projecting from the segments. Tubes 260 have corrugations 268 which facilitate their collapse when compressive load is applied by tightening the fasteners. Note that the segments 256 and 258 are similar to those described above with respect to FIGS. 1 and 2, wherein the arcuate surfaces of the segments have a greater radius of curvature than the pipe elements.

Another example of a spacer for maintaining coupling segments in spaced relation is shown in FIG. 23. Coupling 270 is comprised of segments 272 and 274 having outwardly projecting lugs 266 and 268 positioned in facing relation when the coupling is pre-assembled. The segments are held together by fasteners 280 extending between the lugs. Spacers 282, preferably in the form of block-shaped bodies 284, are positioned between the lugs 266 and 268. The bodies 284 are removable from between the lugs to allow the fasteners to be tightened and draw the segments into engagement with pipe elements being joined.

Bodies 284 may be releasably attached to the segments, for example, held by friction between the lugs 266 and 268. Flexible, resilient materials are particularly advantageous for forming the bodies because bodies made from such materials provide adequate strength and stiffness to maintain the couplings in spaced apart relation during rough handling but may be readily deformed for easy removal as required. If polymer materials are used to form the bodies they may be adhered to the lugs by heat fusing or by adhesives which afford a releasable bond between the bodies and the segments.

FIG. 24 illustrates a non-deformable coupling embodiment 286 that uses spacers 288 to maintain coupling segments 290 and 292 in spaced apart relation so that pipe elements may be inserted between them in the preassembled state shown. Coupling 286 has no notches or other features that provide clearance facilitating inserting pipe elements into end to end relation between the segments, but relies on the spacers to provide sufficient separation for adequate clearance. The spacers 288 may be similar to those described above.

Spacers according to the invention may also be used with various other types of couplings. As shown in FIG. 19, a spacer 288 may be used with the hinged coupling embodiment 192 to keep the lugs 202 and 204 in spaced apart relation so that pipe elements may be inserted. Although a tubular spacer is illustrated, it is understood that any of the spacers described herein are feasible for use with this coupling.

FIG. 25 illustrates an adapter coupling 294 for joining flanged pipe to non-flanged pipe, for example, grooved or plain end. Coupling 294 comprises coupling segments 296



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and 298, each of which has a radially extending flange 300 on one side and a arcuate surface 302 on the opposite side. Segments 296 and 298 are held in spaced apart relation by spacers 304, which may comprise a collapsible tubular spacer 306 or a removable spacer 308, or other types of spacers described herein.

FIG. 26 illustrates further types of spacer embodiments 310 feasible for maintaining coupling segments 312 and 314 in spaced apart relation. Spacers 310 comprise spring elements which deform, preferably substantially elastically, when subjected to a compression force by the fasteners 316. Spring elements may take any of a number of forms, for example, a rubber cylinder 318 or a coil spring 320. The use of spring elements for spacers allows for fine control of the forces required to draw the segments toward one another and also facilitates the reuse of the couplings when the deformation of the spring elements is substantially elastic.

It is anticipated that deformable couplings may also include features such as the tongue and recess disclosed in U.S. Pat. Nos. 6,170,884 and 6,302,450; outlets incorporated within a segment as disclosed in U.S. Pat. No. 3,362,730; plain end couplings that do not use grooves as disclosed in U.S. Pat. Nos. 2,439,979, 3,024,046, 5,911,446 and 6,302,450, all of these patents being hereby incorporated by reference herein.

Deformable mechanical pipe couplings according to the invention provide for rapid and sure installation creating a pipe joint while avoiding the need to partially or totally disassemble and then reassemble the coupling and handle the individual piece parts.

What is claimed is:

1. In combination, interconnectable pipe coupling segments and a pair of pipe elements, each said coupling segment being positionable straddling facing end portions of said pair of pipe elements for securing said pipe elements together in end-to-end relationship, wherein said end portions of said pipe elements have an outer surface of substantially cylindrical profile, said pipe elements each having a circumferential groove in said outer surfaces of said end portions, and wherein each said segment comprises:

first and second arcuate surfaces adapted to contact the outer surfaces of said pipe elements within said circumferential grooves, said arcuate surfaces being in spaced apart relation to one another on opposite sides of said segment, each of said arcuate surfaces subtending an angle of less than 180° and having a radius of curvature greater than the radius of curvature of said outer surfaces of said pipe elements exclusive of said circumferential grooves; and

connection members for adjustably connecting one coupling segment to another, said connection members being adjustably tightenable for drawing said arcuate surfaces of said segments into contact with said outer surfaces of said pipe elements within said circumferential grooves, said segments being deformable upon adjustable tightening of said connection members so as to substantially conform the curvature of said arcuate surfaces to the outer surfaces of said pipe elements within said circumferential grooves.

2. Interconnectable pipe coupling segments according to claim 1, wherein said segments are substantially elastically deformable.

3. Interconnectable pipe coupling segments according to claim 1, wherein said segments are substantially plastically deformable.

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4. Interconnectable pipe coupling segments according to claim 1, wherein said arcuate surfaces project radially inwardly from said segments.

5. Interconnectable pipe coupling segments according to claim 4, wherein said arcuate surfaces are substantially continuous along said segments.

6. Interconnectable pipe coupling segments according to claim 4, wherein one of said arcuate surfaces has a cut-out therein, said cut-out increasing the bending flexibility of said segment.

7. Interconnectable pipe coupling segments according to claim 4, wherein each of said segments comprises first and second sidewalls positioned in spaced relation to one another and extending radially inwardly, said arcuate surfaces being positioned on said sidewalls, said sidewalls defining a channel positioned between said first and second arcuate surfaces.

8. Interconnectable pipe coupling segments according to claim 7, wherein each of said segments includes a concavity positioned between said sidewalls.

9. Interconnectable pipe coupling segments according to claim 1, wherein said arcuate surfaces subtend an angle between about 40° and about 179°.

10. Interconnectable pipe coupling segments according to claim 1, wherein one of said arcuate surfaces has a smaller radius of curvature than the other of said arcuate surfaces.

11. Interconnectable pipe coupling segments according to claim 1, wherein said connection members comprise a pair of projections extending outwardly from the ends of each of said segments, said projections being adapted to receive fasteners for adjustably connecting said segments to one another, said fasteners being adjustably tightenable for drawing said arcuate surfaces of said segments together into engagement with said outer surfaces of said pipe elements.

12. Interconnectable pipe coupling segments according to claim 11, wherein said projections are angularly oriented with respect to one another.

13. Interconnectable pipe coupling segments according to claim 12, wherein said projections are oriented at an angle of up to about 10° with respect to one another.

14. Interconnectable pipe coupling segments according to claim 13, wherein at least one of said projections comprises a lug having an aperture therethrough adapted to receive said fastener.

15. Interconnectable pipe coupling segments according to claim 1, wherein said arcuate surfaces comprise at least one tooth projecting substantially radially inwardly from said segments, said tooth gripping said outer surfaces of said pipe elements and providing mechanical restraint.

16. Interconnectable pipe coupling segments according to claim 1, further comprising a plurality of teeth positioned on said segments adjacent to said arcuate surfaces, said teeth projecting radially inwardly for engagement with said end portions of said pipe elements when said segments are deformed so as to substantially conform to the curvature of said outer surfaces of said pipe elements.

17. Interconnectable pipe coupling segments according to claim 1, wherein each of said segments further comprises a pair of angularly oriented surfaces positioned adjacent to each of said connection members, said angularly oriented surfaces on each of said segments having opposite slopes, said angularly oriented surfaces on one said segment being in facing relation with said angularly oriented surfaces on another said segment, said angularly oriented segments engaging one another when said segments are drawn together and causing said segments to rotate relatively to one another about an axis substantially perpendicular to said pipe elements.

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18. Interconnectable pipe coupling segments according to claim 1, wherein one of said segments has a section of increased bending flexibility positioned between said connection members, said section facilitating deforming of said segment for conforming the curvature of said segment substantially to the outer surfaces of said pipe elements.

19. Interconnectable pipe coupling segments according to claim 18, wherein said section of increased bending flexibility is achieved by reducing the area moment of inertia of said segment between said connection members.

20. Interconnectable pipe coupling segments according to claim 18, wherein said section of increased bending flexibility comprises a cut-out positioned in said segment.

21. In combination, a pair of pipe elements and a pipe coupling, said pipe coupling being positionable straddling facing end portions of said pair of pipe elements for securing said pipe elements together in end-to-end relationship, wherein said end portions of said pipe elements have an outer surface of substantially cylindrical profile, said pipe elements each having a circumferential groove in said outer surfaces of said end portions, and wherein said pipe coupling comprises:

first and second coupling segments, each said coupling segment having first and second arcuate surfaces adapted to contact the outer surfaces of said pipe elements within said circumferential grooves, said arcuate surfaces being in spaced apart relation to one another on opposite sides of said segment, each of said arcuate surfaces subtending an angle of less than 180° and having a radius of curvature greater than the radius of curvature of said outer surfaces of said pipe elements exclusive of said circumferential grooves; and

said first and second coupling segments being connected to one another end to end surrounding a central space, each said coupling segment having connection members adjustably connecting one said coupling segment to another, said connection members being adjustably tightenable for drawing said arcuate surfaces of said segments into contact with said outer surfaces of said pipe elements within said circumferential grooves when said pipe elements are inserted into said central space, said segments being deformable upon adjustable tightening of said connection members so as to substantially conform the curvature of said arcuate surfaces to the outer surfaces of said pipe elements within said circumferential grooves.

22. A pipe coupling according to claim 21, wherein said segments are substantially elastically deformable.

23. A pipe coupling according to claim 21, wherein said segments are substantially plastically deformable.

24. A pipe coupling according to claim 21, wherein said arcuate surfaces project radially inwardly from said segments.

25. A pipe coupling according to claim 24, wherein each of said segments comprises first and second sidewalls positioned in spaced relation to one another and extending radially inwardly, said arcuate surfaces being positioned on said sidewalls, said sidewalls defining a channel positioned between said first and second arcuate surfaces.

26. A pipe coupling according to claim 25, wherein each of said segments includes a concavity positioned between said sidewalls.

27. A pipe coupling according to claim 25, further comprising a flexible, resilient seal, said seal being a substantially circular ring having an inner diameter sized to receive said pipe elements, said seal being positioned within said channels of said first and second coupling segments, said seal having an outer diameter sized to support said first and second coupling

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segments in spaced apart relation while said first and second segments are attached to one another thereby allowing said pipe elements to be inserted between said first and second coupling segments into said end-to-end relationship, said seal being deformable when said connection members are adjustably tightened to draw said arcuate surfaces together and conform the curvature of said segments substantially to the outer surfaces of said pipe elements within said circumferential grooves.

28. A pipe coupling according to claim 21, wherein said arcuate surfaces subtend an angle between about 40° and about 179°.

29. A pipe coupling according to claim 21, wherein said arcuate surfaces include first and second notches positioned at opposite ends thereof, said notches providing clearance for insertion of said pipe elements between said first and second segments.

30. A pipe coupling according to claim 21, wherein one of said segments has a section of increased bending flexibility positioned between said connection members, said section facilitating deforming of said segment for substantially conforming the curvature of said segment to the outer surfaces of said pipe elements.

31. A pipe coupling according to claim 30, wherein said section of increased bending flexibility is achieved by reducing the area moment of inertia of said segment between said connection members.

32. A pipe coupling according to claim 21, wherein one of said arcuate surfaces on each of said first and second segments has a smaller radius of curvature than the other of said arcuate surfaces on said first and second segments.

33. A pipe coupling according to claim 21, wherein said connection members comprise a pair of projections extending outwardly from the ends of each of said segments, said projections being adapted to receive fasteners for adjustably connecting said segments to one another, said fasteners being adjustably tightenable for drawing said arcuate surfaces of said segments together into engagement with said outer surfaces of said pipe elements.

34. A pipe coupling according to claim 33, wherein said projections are angularly oriented with respect to one another.

35. A pipe coupling according to claim 34, wherein said projections are oriented at an angle of up to about 10° with respect to one another.

36. A pipe coupling according to claim 33, wherein said fasteners comprise nuts threaded onto bolts, said bolts having threads that are deformed so as to temporarily hold said nuts in a predetermined position compatible with the spaced apart relationship of said first and second coupling segments, said nuts being adjustably tightenable from said predetermined position for drawing said arcuate segments into engagement with said outer surfaces of said pipe elements.

37. A pipe coupling according to claim 36 wherein said threads are deformed by staking said threads.

38. A pipe coupling according to claim 21, further comprising a plurality of teeth positioned on said first and second segments adjacent to said arcuate surfaces, said teeth projecting radially inwardly for engagement with said end portions of said pipe elements when said segments are deformed so as to substantially conform to the curvature of said outer surfaces of said pipe elements.

39. A pipe coupling according to claim 21, wherein each of said first and second segments further comprises a pair of angularly oriented surfaces positioned adjacent to each of said connection members, said angularly oriented surfaces on one said segment being in facing relation with said angularly oriented surfaces on another said segment, said angularly



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oriented surfaces engaging one another when said segments are drawn together and causing said segments to move relatively to one another and engage said arcuate surfaces with said outer surfaces of said pipe elements.

40. A pipe coupling according to claim 39, wherein said angularly oriented surfaces on each of said segments have opposite slopes, said angularly oriented surfaces engaging one another when said segments are drawn together and causing said segments to rotate relatively to one another about an axis substantially perpendicular to said pipe elements.

41. In combination, a pair of pipe elements and a pipe coupling, said pipe coupling being positionable straddling facing end portions of said pair of pipe elements for securing said pipe elements to one another in end-to-end relation, wherein said end portions of said pipe elements have an outer surface of substantially cylindrical profile, said pipe elements each having a circumferential groove in said outer surfaces of said end portions, said coupling comprising:

a plurality of coupling segments, each said coupling segment having first and second arcuate surfaces adapted to contact the outer surfaces of said pipe elements within said circumferential grooves, said arcuate surfaces being in spaced apart relation to one another on opposite sides of each said segment, each of said arcuate surfaces subtending an angle of less than 180° and having a radius of curvature greater than the radius of curvature of said outer surfaces of said pipe elements exclusive of said circumferential grooves; and

said coupling segments being connected to one another end to end surrounding a central space, each said coupling segment having connection members adjustably connecting one coupling segment to another, said connection members being adjustably tightenable for drawing said arcuate surfaces of said segments into contact with said outer surfaces of said pipe elements within said circumferential grooves when said pipe elements are inserted into said central space, said segments being deformable upon adjustable tightening of said connection members so as to substantially conform the curvature of said arcuate surfaces to the outer surfaces of said pipe elements within said circumferential grooves.

42. A pipe coupling according to claim 41, comprising four of said coupling segments.

43. A pipe coupling according to claim 41, further comprising a flexible, resilient seal, said seal being a substantially circular ring having an inner diameter sized to receive said pipe elements, said seal being positioned between said arcuate

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ate surfaces of said segments, said seal having an outer diameter sized to support said coupling segments in spaced apart relation while said piping segments are attached to one another thereby allowing said pipes to be inserted between said coupling segments into said end-to-end relationship, said seal being deformable when said connection members are adjustably tightened to draw said arcuate surfaces toward one another and substantially conform the curvature of said segments to the outer surfaces of said pipes within said circumferential grooves.

44. In combination, a pipe coupling and a pair of pipe elements, said pipe coupling being positioned straddling facing end portions of said pipe elements for securing said pipe elements to one another in end-to-end relationship, wherein end portions of said pipe elements have an outer surface of substantially cylindrical profile, said pipe elements each having a circumferential groove in said outer surfaces of said end portions, said coupling comprising:

first and second coupling segments, each said coupling segment having first and second arcuate surfaces adapted to contact the outer surfaces of said pipe elements within said circumferential grooves, said arcuate surfaces being in spaced apart relation to one another on opposite sides of said segment, each of said arcuate surfaces subtending an angle of less than 180° and having a radius of curvature greater than the radius of curvature of said outer surfaces of said pipe elements exclusive of said circumferential grooves; and

said first and second coupling segments being connected to one another end to end surrounding said facing end portions of said pipe elements, each said coupling segment having connection members adjustably connecting one said coupling segment to another, said connection members being adjustably tightenable for drawing said arcuate surfaces of said segments into contact with said outer surfaces of said pipe elements within said circumferential grooves, said segments being deformable upon adjustable tightening of said connection members so as to substantially conform the curvature of said arcuate surfaces to the outer surfaces of said pipe elements within said circumferential grooves.

45. A combination according to claim 44, wherein one of said pipe elements has a smaller diameter than the other of said pipe elements, and one of said arcuate surfaces has a smaller radius of curvature than the other of said arcuate surfaces.

\* \* \* \* \*

## Exhibit 3



FIGURE 7402

**Slide** **LOCK**

Rigid Coupling

Patents Pending



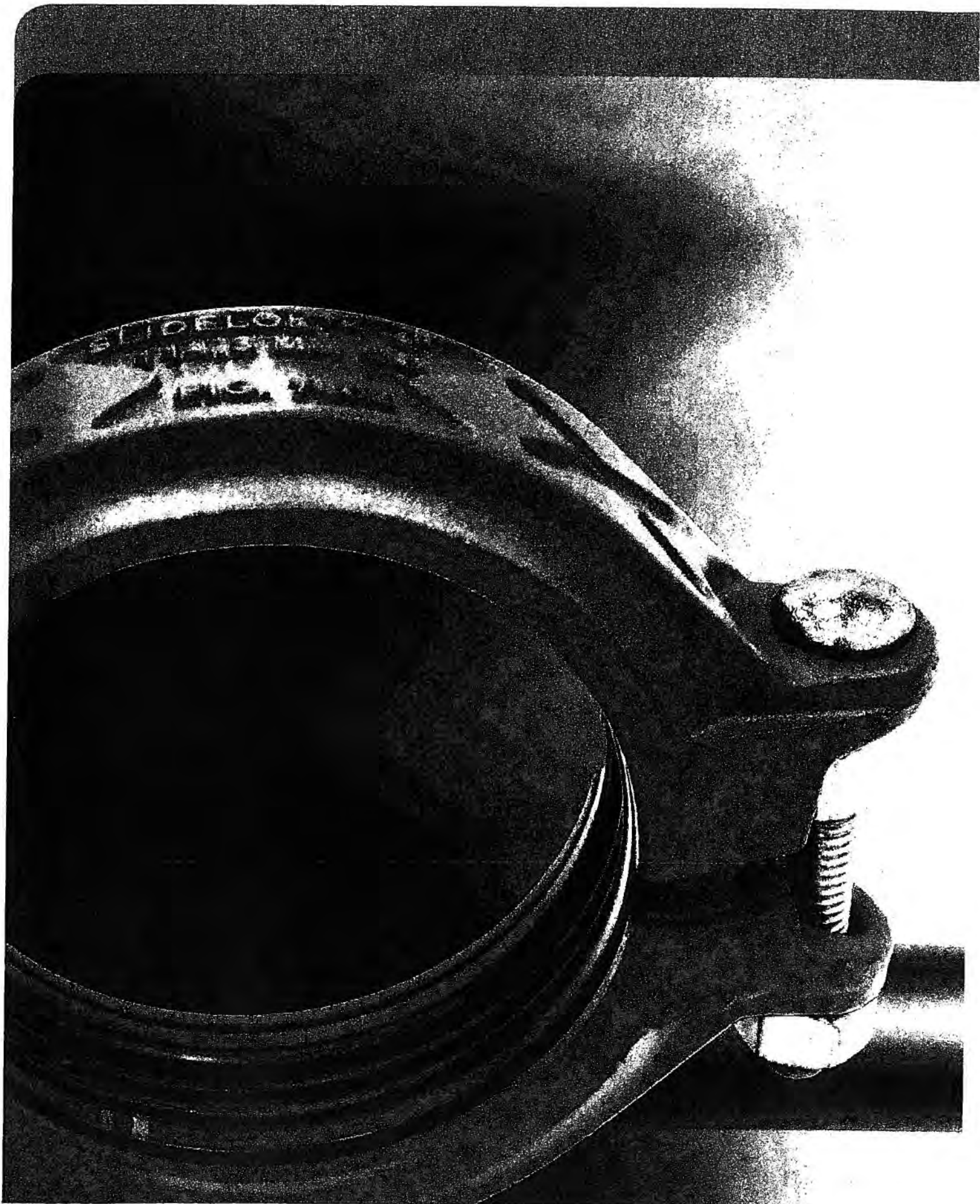
B U I L D I N G   C O N N E C T I O N S   T H A T   L A S T



October 2012

For the most current product pricing  
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our website at [www.anvilint.com](http://www.anvilint.com)







**Anvil International, a world-wide leader in mechanical piping products, is pleased to introduce the most advanced ready for installation coupling available, manufactured with pride in the U.S.A.**

Utilizing advanced engineering, Anvil introduces the SlideLOK™ rigid coupling. The coupling is truly unique and offers the safest procedure for installation. There are no loose parts.

The coupling is installed by simply lubricating the exposed gasket surfaces, and then sliding the coupling completely over the pipe end. This method eliminates the possibility of the coupling falling off of the pipe as the entire SlideLOK coupling rests on the pipe end. Installation is completed by bringing the adjacent component up to the pipe end, and sliding the coupling back over both pipe ends. The joint is finished by tightening the two bolts with an impact wrench or socket wrench.

The SlideLOK coupling utilizes a tongue and groove design to eliminate gasket pinching. When the coupling is tightened bolt pad to bolt pad, the joint is completed. Simple and safe.

**Specification Information:**

Ready for installation couplings shall be Gruvlok #7402 in sizes 2" through 8". Gaskets shall be type "EP" EPDM for water service (-40°F to +250°F), and "T" Nitrile for petroleum services (-20°F to +180°F).

The SlideLOK coupling features our new pressure-responsive gasket. The patent-pending gasket is the result of state-of-the-art CAD generated design. This innovative gasket features four separate sealing contacts to seal the gasket on the pipe. The gasket employs a unique leading edge to eliminate gasket pinching and patented gasket lips for easy slide action. The gasket is our EP formulated EPDM gasket and carries a 250°F temperature rating.

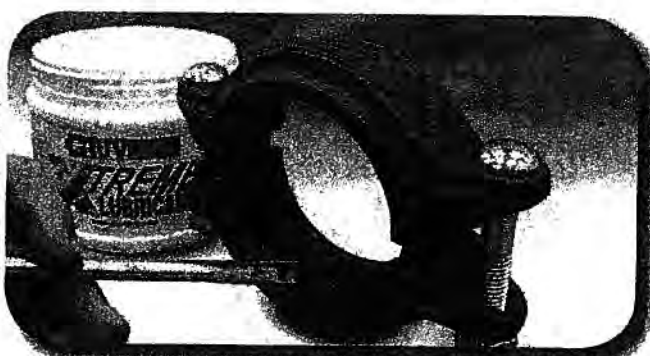
**SlideLOK®**



# Installation

## Ready for installation - right out of the box

Do not disassemble the SlideLOK™ Coupling. The Figure 7402 coupling is ready for installation. The bolt and gasket do not need to be removed.



### Step 1 – Pipe Preparation

Pipe ends are to be rolled or cut grooved according to Anvil specifications. The pipe end must be smooth and free from metal burrs or projections.

### Step 2 – Gasket Preparation

Ensure the gasket is suitable for the intended application by referring to the Anvil gasket compatibility chart. Apply a light coating of Gruvlok® Xtreme™ Lubricant to exposed gasket surfaces.

### Step 3 – Assembly

The SlideLOK Figure 7402 may be installed by one of two methods. The preferred method depends on the type of pipe components being joined and their orientation. Please review both methods before installing.

#### Method #1

Slide the SlideLOK coupling completely over the grooved pipe end. This will allow a clear and un-obstructed view of the pipe for correct alignment.



**A.** Slide the coupling on the pipe past the groove. The bolts and nuts can be hand tightened to position the coupling in place.

**B.** Align the mating pipe end. Align the two adjoining pipes together.



**C.** Slide the coupling back over the grooves so that the coupling keys are located over the respective grooves on both pipe ends.

**D.** Follow the instructions on fastening the coupling as shown in Step 4.

METHOD

#1

Slide



## Method #2

Slide the SlideLOK™ coupling half way onto the pipe end or fitting. This will better accommodate fitting, and valve accessories during installation.



A. Slide the coupling on the fitting so that the groove and keys are aligned.

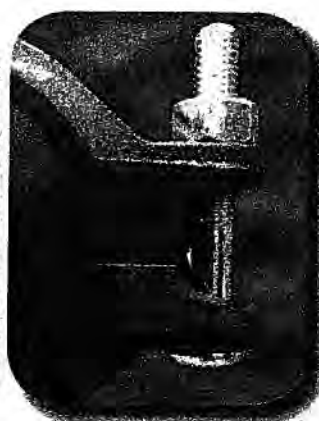


B. Bring the pipe end or fitting towards the coupling and insert so that the groove and coupling keys are aligned.

C. Hand tighten the nuts to correctly position the couplings keys over the respective grooved ends

D. Follow the instructions on fastening the coupling as shown in Step 4.

METHOD  
#2



### Step 4 – Final Assembly

The SlideLOK coupling is designed to achieve pad to pad (metal-to-metal contact) using either an impact wrench\* or wrench. The intended torque range for the coupling is located in the Table 1. Securely tighten nuts alternately and equally until the housing halves are in metal-to-metal contact.

### Step 5 – Final Inspection

Ensure the coupling is properly aligned in the grooves and the housing halves are in metal-to-metal contact, depicted in the picture above.

Table 1 – Torque Ranges

Sizes	Torque (Ft-Lbs)
2" - 4"	80 - 100
5" - 6"	100 - 130

\* **CAUTION:** When using an impact wrench, verify that the output of the torque wrench is within the required torque range. It is recommended that a torque wrench be used for accurate assembly in order to obtain specified performance.

**Slide**



# Re-Installation

## Reinstallation of the Figure 7402 SlideLOK™ Coupling

The SlideLOK coupling is designed to be installed in the ready for installation assembly position once. After the initial assemble the following steps are to be taken to re-install the Fig. 7402 SlideLOK coupling.

### Step 1 – De-pressurize the System

De-pressurize the system before removing the SlideLOK Coupling. Dis-assemble the couplings by removing the nuts, bolts and gasket from the housing halves. A wrench is required to overcome the epoxy used to secure the nuts on the bolts.



### Step 2 – Pipe Preparation

Pipe ends are to be rolled or cut grooved according to Anvil specifications. The pipe end must be smooth and free from metal burrs or projections.



### Step 3 – Gasket Preparation

Ensure the gasket is suitable for the intended application by referring to the Anvil gasket compatibility chart. A light coating of Gruvlok® XTreme™ lubricant must be applied to the gasket prior to installation.

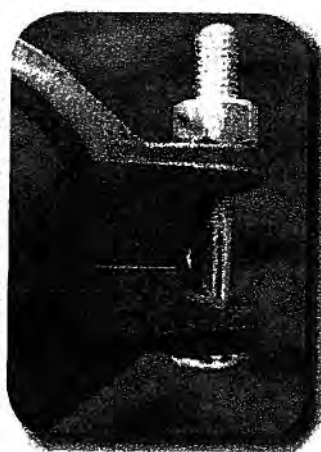
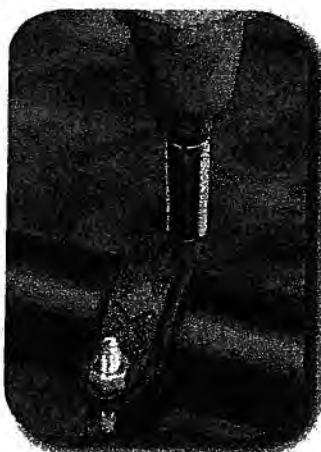
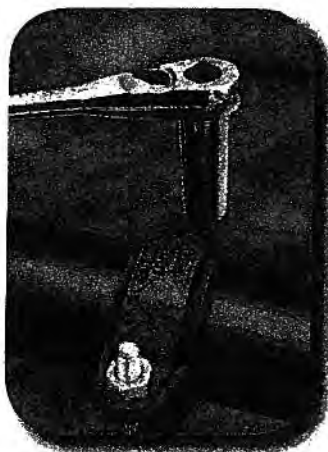


### Step 4 – Pipe Alignment and Gasket Installation

Slide the gasket onto the pipe then align the two pipe ends together. Pull the gasket into position, centering it between the grooves on each pipe. Gasket should not extend into the groove on either pipe.

### Step 5 – Housing Assembly

Place each housing halves on the pipe making sure the housing key fits into the groove. Be sure that the tongue and recess portions of the housing mate properly. Insert the bolts.



### Step 6 – Final Assembly

The SlideLOK coupling is designed to achieve pad to pad (metal-to-metal contact) using either an impact wrench\* or wrench. The intended torque range for the coupling is located in the Table 1. Securely tighten nuts alternately and equally until the housing halves are in metal-to-metal contact.

### Step 7 – Final Inspection

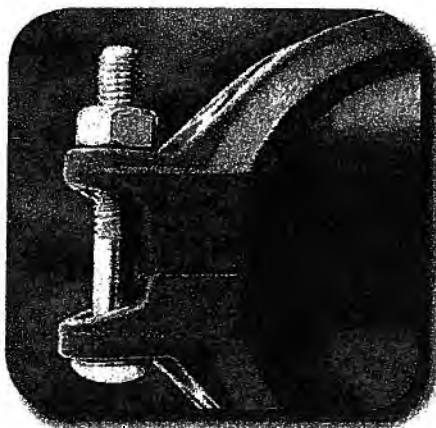
Ensure the coupling is properly aligned in the grooves and the housing halves are in metal-to-metal contact, depicted in the picture above.

Table 1 – Torque Ranges

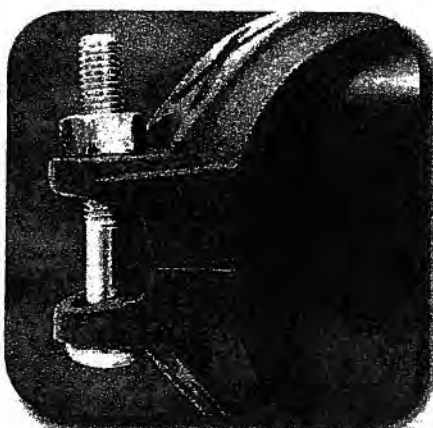
Sizes	Torque (Ft-Lbs)
2" - 4"	80 - 100
5" - 6"	100 - 130

**\* CAUTION:** When using an impact wrench, verify that the output of the torque wrench is within the required torque range. It is recommended that a torque wrench be used for accurate assembly in order to obtain specified performance.

### Incorrect Installation Examples



Low Torque or Out of Groove



Excess Torque or Shallow Groove Dimension







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\*Inventory varies at locations

BUILDING CONNECTIONS THAT LAST



